JS 44 (Rev. 3/99)

CIVIL COVER SHEET

APPENDIX H

The JS-44 civil cover sheet and the information contained herein neither replace nor supplement the filing and service of pleadings or other papers as required by law, except as provided by local rules of court. This form, approved by the Judicial Conference of the United States in September 1974, is required for the use of the Clerk of Court for the purpose of initiating the civil docket sheet. (SEE INSTRUCTIONS ON THE REVERSE OF THE FORM.)

	(a) PLAINTIFF				DEFENDANTS	3					
	GRACO CHILDREN'S PRODUCTS, INC. 150 Oaklands Boulevard Exton, PA 19341				FISHER-PRICE, INC. 636 Girard Avenue East Aurora, NY 14052						
	(b) County of Reside	ent of First Listed Plaintiff(EXCEPT IN	U.S. PLAINTIFF C	ASES)		(IN	U.S. PI MNATIC	Defendant LAINTIFF CASES C ON CASES, USE THE L		√ OF	
	Lynn E. Rzor Ballard Spahi	n Name, Address, and Telephon nea, Esquire 21 or Andrews & Ingersoll, L St., 51 st Fl., Phila., PA 19	5-864-8109 LP		Attorneys (If Kr	own)					
I.	BASIS OF JURISDI	CTION (Place an "X" in One Bo	x Only)		ZENSHIP OF PR Diversity Cases Only)		PARTI	ES (Place an "X" in Or and One Box fo			
]1	U.S. Government Plaintiff		a Pany)	Citizen of		PTF	DEF □ 1	Incorporated or Princi of Business In This S		PTF □4	DEF □ 4
]2	U.S. Government Defendant	☐ 4 Diversity (Indicate Citizenship of	Parties in Item III)		Another State	□ 2 □ 2	□ 2	Incorporated and Print Of Business in Anot			□ 5
37	NATURE OF SUIT	(Place an "X" in One Box Only)		Citizen or C Foreign C	Subject of a Country	□ 3	☐ 3	Foreign Nation		□ 6	□ 6
¥ .	CONTRACT		RTS		FORFEITURE/PE	NALTY	B	ANKRUPTCY	OTHE	RSTATUT	ES
	10 Insurance 20 Marine 31 Miller Act 40 Negotiable Instrument 51 Recovery of Overpayment &Enforcement of Judgment 51 Medicare Act 52 Recovery of Defaulted 51 Student Loans (Excl. Veterans) 53 Recovery of Overpayment of Veteran's Benefits 60 Stockholders' Suits 90 Other Contract 90 Other Contract Contract Product Liability REAL PROPERTY 10 Land Condemnation 50 Foreclosure 30 Rent Lease & Ejectment 45 Tort Product Liability 41 Other Real Property	PERSONAL INJURY 310 Airplane 315 Airplane Product Liability 320 Assault, Libet & Slander 330 Federal Employers' Liability 340 Marine 345 Marine Product Liability 350 Motor Vehicle 355 Motor Vehicle 360 Officer Personal Injury CIVIL RIGHTS 441 Voting 442 Employment 443 Housing/ Accommodations 444 Welfare 440 Other Civil Rights	PERSONAL INJURY 362 Personal Injury Med. Malpract 365 Personal Injury Product Liabili 368 Asbestos Perso Injury Product Liability PERSONAL PROPEF 370 Other Fersonal 371 Truth in Lendi 380 Other Personal Property Dams Property Dams Product Liabil PRISONER PETT 510 Motions to Va Sentence Habeas Corpus: 530 General 535 Death Penalty 540 Mandamus & Covil Rights 550 Civil Rights 555 Prison Conditi	r. ice /- ity snal RTY ng ge spe stity TIONS cute	□ 610 Agriculture □ 620 Other Food & C □ 625 Drug Related Sc of Property 21 U □ 630 Liquor Laws □ 640 R.R. & Truck □ 650 Airline Regs. □ 660 Occupational Sc □ 660 Occupational Sc □ 710 Fair Labor Stan Act □ 720 Labor/Mgmt. R □ 730 Labor/Mgmt. R □ 740 Railway Labor □ 790 Other Labor Labor □ 791 Empl. Ret. Inc. Security Act	afety/Health afety/Health dards clations eporting et Act	423 PF 520 830 840 861 862 863 405(G	Appeal Individual 28 USC 157 ROPERTY RIGHTS Copyrights Copyrights Patent Trademark OCIAL SECURITY HIA (1395ff) Black Lung (923) DIWC/DIWW Company Co	410 / 410 / 410 430 E 450 E 45	itate Reapporticution to the continuation to t	king Rates/etc. enced and zations se modifies/ enge Actions ts filization Act Matters ion Act t Under
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VII.	REQUESTED IN COMPLAINT	Brief description of cause: Patent Infringement CHECK IF THIS IS A C UNDER F.R.C.P. 23 de		nctive	DEMAND S			CHECK YES on JURY DEMAN			olaint:] No
VIII.	. RELATED CASE(S) IF ANY	(See instructions):	JUDG	€				DOCKET NUMBE	R		
	tober 27, 2005		SIGNATURE	GF ATTORN	EY OF RECORD	gene	a_		***************************************		
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INSTRUCTIONS FOR ATTORNEYS COMPLETING CIVIL COVER SHEET FORM JS-44

Authority For Civil Cover Sheet

The JS-44 civil cover sheet and the information contained herein neither replaces nor supplements the filings and service of pleading or other papers as required by law, except as provided by local rules of court. This form, approved by the Judicial Conference of the United States in September 1974, is required for the use of the Clerk of Court for the purpose of initiating the civil docket sheet. Consequently, a civil cover sheet is submitted to the Clerk of Court for each civil complaint filed. The attorney filing a case should complete the form as follows:

- I. (a) Plaintiffs-Defendants. Enter names (last, middle initial) of plaintiff and defendant. If the plaintiff or defendant is a government agency, use only the full name or standard abbreviations. If the plaintiff or defendant is an official within a government agency, identify first the agency and then the official, giving both name and title.
- (b) County of Residence. For each civil case filed, except U.S. plaintiff cases, enter the name of the county where the first listed plaintiff resides at the time of filing. In U.S. plaintiff cases, enter the name of the county in which the first listed defendant resides at the time of filing. (NOTE: In land condemnation cases, the county of residence of the "defendant" is the location of the tract of land involved.)
- (c) Attorneys. Enter the firm name, address, telephone number, and attorney of record. If there are several attorneys, list them on an attachment, noting in this section "(see attachment)".
- II. Jurisdiction. The basis of jurisdiction is set forth under Rule &(a), F.R.C.P., which requires that jurisdictions be shown in pleadings. Place an "X" in one of the boxes. If there is more than one basis of jurisdiction, precedence is given in the order shown below.

United States plaintiff. (1) Jurisdiction based on 28 U.S.C. 1345 and 1348. Suits by agencies and officers of the United States, are included here.

United States defendant. (2) When the plaintiff is suing the United States, its officers or agencies, place an "X" in this box.

Federal question. (3) This refers to suits under 28 U.S.C. 1331, where jurisdiction arises under the Constitution of the United States, an amendment to the Constitution, an act of Congress or a treaty of the United States. In cases where the U.S. is a party, the U.S. plaintiff or defendant code takes precedence, and box 1 or 2 should be marked.

Diversity of citizenship. (4) This refers to suits under 28 U.S.C. 1332, where parties are citizens of different states. When Box 4 is checked, the citizenship of the different parties must be checked. (See Section III below; federal question actions take precedence over diversity cases.)

- III. Residence (citizenship) of Principal Parties. This section of the JS-44 is to be completed if diversity of citizenship was indicated above. Mark this section for each principal party.
- IV. Nature of Suit. Place an "X" in the appropriate box. If the nature of suit cannot be determined, be sure the cause of action, in Section IV below, is sufficient to enable to deputy clerk or the statistical clerks in the Administrative Office to determine the nature of suit. If the cause fits more than one nature of suit, select the most definitive.
- V. Origin. Place an "X" in one of the seven boxes.

Original Proceedings. (1) Cases which originate in the United States district courts.

Remove from State Court. (2) Proceedings initiated in state courts may be removed to the district courts under Title 28 U.S.C., Section 1441. When the petition for removal is granted, check this box.

Remanded from Appellate Court. (3) Check this box for cases remanded to the district court for further action. Use the date of remand as the filing date.

Reinstated or Reopened. (4) Check this box for cases reinstated or reopened in the district court. Use the reopening date as the filing date.

Transferred from Another District. (5) For cases transferred under Title 28 U.S.C. Section 1404(a) Do not use this for within district transfers or multidistrict litigation transfers.

Multidistrict Litigation. (6) Check this box when a multidistrict case is transferred into the district under authority of Title 28 U.S.C. Section 1407. When this box is checked, do not check (5) above.

Appeal to District Judge from Magistrate Judgment. (7) Check this box for an appeal from a magistrate judge's decision.

- VI. Cause of Action. Report the civil statute directly related to the cause of action and give a brief description of the cause.
- VII. Requested in Complaint. Class Action. Place an "X" in this box if you are filing a class action under Rule 23, F.R.Cv.P.

Demand. In this space enter the dollar amount (in thousands of dollars) being demanded or indicate other demand such as a preliminary injunction.

Jury Demand. Check the appropriate box to indicate whether or not a jury is being demanded.

VIII. Related Cases. This section of the JS-44 is used to reference related pending cases if any. If there are related pending cases, insert the docket numbers and the corresponding judge names for such cases.

Date and Attorney Signature. Date and sign the civil cover sheet.

IN THE UNITED STATES DISTRICT COURT FOR THE EASTERN DISTRICT OF PENNSYLVANIA

CASE MANAGEMENT TRACK DESIGNATION FORM

GRACO CI	HILDREN'S PRO	DDUCTS, INC.	: CIVIL ACTION	
	v.		· :	
FISHER-PRICE, INC. : NO.				
plaintiff sha filing the c reverse side said design on the plain	all complete a ca complaint and ser e of this form.) ation, that defend ntiff and all other	se Management Track Des ve a copy on all defendar In the event that a defendar lant shall, with its first apport	lay Reduction Plan of this court, counsel for signation Form in all civil cases at the time of ints. (See § 1:03 of the plan set forth on the int does not agree with the plaintiff regarding earance, submit to the clerk of court and serve int track designation form specifying the track igned.	
SELECT (ONE OF THE F	OLLOWING CASE MAN	NAGEMENT TRACKS:	
(a)	Habeas Corpus	s – Cases brought under 28	U.S.C. §2241 through §2255.	
(b)			of a decision of the Secretary of atiff Social Security Benefits.	
(c)	Arbitration – C Civil Rule 53.2		nated for arbitration under Local	
(d)	Asbestos – Ca from exposure		ersonal injury or property damage	
(e) Special Management – Cases that do not fall into tracks (a) through (d) that are commonly referred to as complex and that need special or intense management by the court. (See reverse side of this form for a detailed explanation of special management cases.)				
(f) Standard Management - Cases that do not fall into any one of the other tracks				
10/27	7/2005	Lynn E. Rzonca	Graco Children's Products, Inc.	
	ate	Attorney-at-law	Attorney for	
215-86	64-8109	215-864-9278	rzoncal@ballardspahr.com	
Telephone		FAX Number	E-Mail Address	

UNITED STATES DISTRICT COURT

FOR THE EASTERN DISTRICT OF PENNSYLVANIA - DESIGNATION FORM to be used by counsel to indicate the course of assignment to appropriate calendar.	ategory of th	e case for the			
Address of Plaintiff: 150 Oaklands Boulevard, Exton, PA 19341	**************************************				
Address of Defendant: 636 Girard Avenue, East Aurora, NY 14052					
Place of Accident, incident or Transaction:(Use Reverse Side For Additional Space)					
Does this civil action involve a nongovernmental corporate party with any parent corporation and any publicly held corporation owning 10%					
(Attach two copies of the Disclosure Statement Form in accordance with Fed.R.Civ.P. 7.1(a))	Yes⊠	No 🔲			
Does this case involve multidistrict litigation possibilities?	Yes□	No 🗵			
RELATED CASE IF ANY					
Case Number: Judge Date Terminated:					
Civil cases are deemed related when yes is answered to any of the following questions:					
 Is this case related to property included in an earlier numbered suit pending or within one year previously terminated action in this court? 	Yes 🗌	No 🛛			
2. Does this case involve the same issue of fact or grow out of the same transaction as a prior suit pending or within one year previously terminated action in this court?	Yes 🗌	No 🗵			
3. Does this case involve the validity or infringement of a patent already in suit or any earlier numbered case pending or within one year previously terminated action in this court?	Yes 🗌	No 🛛			
A. Federal Questions Cases: 1.					
(Check appropriate category) I. Lynn E. Rzonca, counsel of record do hereby certify:					
Pursuant to Local Civil Rule 53.2, Section 3(c)(2), that, to the best of my knowledge and belief, the damages reaction case exceed the sum of \$150,000.00 exclusive of interest and costs;	ecoverable i	n this civil			
Relief other than monetary damages is sought.					
DATE: October 27, 2005 Attorney-at-Law 86747 Attorney l.D. #					
NOTE: A trial de novo will be a trial by jury only if there has been compliance with F.R.C.P. 3					
I certify that, to my knowledge, the within case is not related to any case now pending or within one year previously this court except as noted above.	terminatio	n action in			
DATE: October 27, 2005 Synn 6. Spanea _	86747				
. * ()					

IN THE UNITED STATES DISTRICT COURT FOR THE EASTERN DISTRICT OF PENNSYLVANIA

Defendant.)
FISHER-PRICE, INC.,)
v.	Jury Trial Demanded
Plaintiff,	Civil Action No
GRACO CHILDREN'S PRODUCTS, INC.,)

Plaintiff Graco Children's Products, Inc. ("Graco") complains of defendant Fisher-Price, Inc. ("Fisher-Price") as follows:

Nature of the Action

1. This is an action for injunctive relief and damages arising out of defendant Fisher-Price's infringement of two United States patents owned by Graco.

Jurisdiction and Venue

2. This action arises under the patent laws of the United States, 35 U.S.C. § 101 et seq. This Court has subject matter jurisdiction pursuant to 28 U.S.C. §§ 1331 (federal question jurisdiction) and 1338(a) (original jurisdiction under patent laws). Venue lies in the Eastern District of Pennsylvania under 28 U.S.C. §§ 1391 and 1400.

The Parties

3. Graco is a Delaware corporation with a place of business in Exton,
Pennsylvania. Graco is a leading manufacturer of juvenile products, including children's
swings, strollers, infant car seats and carriers, high chairs, mobiles, playards, and cribs.

4. On information and belief, Fisher-Price is a Delaware corporation with its principal place of business in East Aurora, New York. Fisher-Price sells children's products, including children's swings, to retailers throughout the United States, including to retailers and consumers in this District.

Fisher-Price's Acts of Infringement

- 5. On June 11, 1996, the United States Patent and Trademark Office issued United States Patent No. 5,525,113 entitled "Open Top Swing & Control" ("the '113 Patent"). A copy of the '113 Patent is attached hereto as Exhibit A.
- 6. At all relevant times, Graco has owned all right, title, and interest in the '113 Patent.
- 7. Fisher-Price was or is making, using, selling, or offering for sale children's swings that infringe one or more claims of the '113 Patent, including: the Deluxe Flutterbye Dreams Swing; the Flutterbye Dreams Swing; the Smart Stages 3 in 1 Rocker Swing; the Magical Mobile Swing; the Deluxe Quick Response Swing; the Deluxe Smart Response Swing; and the Peaceful Time Open Top Swing.
- 8. On information and belief, Fisher-Price was or is selling or offering for sale in this District children's swings, including: the Deluxe Flutterbye Dreams Swing; the Flutterbye Dreams Swing; the Smart Stages 3 in 1 Rocker Swing; the Magical Mobile Swing; the Deluxe Quick Response Swing; the Deluxe Smart Response Swing; and the Peaceful Time Open Top Swing
- 9. On May 24, 2005, the United States Patent and Trademark Office issued United States Patent No. 6,896,624 entitled "Foldable Swing Having Rotatable Handle" (the '624 Patent"). A copy of the '624 Patent is attached as Exhibit B.

- 10. At all relevant times Graco has owned all right, title, and interest in and to the '624 Patent.
- 11. Fisher-Price was or is making, using, selling, or offering for sale a children's swing called the "Fisher-Price Aquarium Take-Along Swing" that infringes one or more claims of the '624 Patent.
- 12. On information and belief, Fisher-Price was or is selling or offering for sale its Fisher-Price Aquarium Take-Along Swing in this District.

COUNT I—PATENT INFRINGEMENT The '113 Patent

- 13. Graco incorporates by reference each of the allegations of the foregoing paragraphs as if fully stated herein.
- 14. Fisher-Price has infringed one or more claims of the '113 Patent by making, using, selling, offering for sale, and/or importing into the United States children's swings, including: the Deluxe Flutterbye Dreams Swing; the Flutterbye Dreams Swing; the Smart Stages 3 in 1 Rocker Swing; the Magical Mobile Swing; the Deluxe Quick Response Swing; the Deluxe Smart Response Swing; and the Peaceful Time Open Top Swing.
 - 15. Graco has been injured by Fisher-Price's infringement of the '113 Patent.
- 16. Graco will continue to be injured by Fisher-Price's infringement of the '113Patent unless the Court enjoins Fisher-Price from further infringement.
- 17. Fisher-Price's infringement has caused and will continue to cause Graco irreparable harm for which there is no adequate remedy at law.
 - 18. Fisher-Price's infringement has been with full knowledge of the '113 Patent.

COUNT II – PATENT INFRINGEMENT The '624 Patent

- 19. Graco incorporates by reference each of the allegations of the foregoing paragraphs as if fully stated herein.
- 20. Fisher-Price has infringed one or more claims of the '624 Patent by making, using, selling, offering for sale, and/or importing into the United States children's swings, including the Fisher-Price Aquarium Take-Along Swing.
 - 21. Graco has been injured by Fisher-Price's infringement of the '624 Patent.
- 22. Graco will continue to be injured by Fisher-Price's infringement of the '624 Patent unless the Court enjoins Fisher-Price from further infringement.
- 23. Fisher-Price's infringement has caused and will continue to cause Graco irreparable harm for which there is no adequate remedy at law.
 - 24. Fisher-Price's infringement has been with full knowledge of the '624 Patent.

PRAYER FOR RELIEF

WHEREFORE, Graco prays that this honorable Court:

- A. Enter judgment against Fisher-Price for infringement of the '113 and '624 Patents;
- B. Issue an Order permanently enjoining Fisher-Price, its officers, directors, agents, servants, employees, attorneys, subsidiaries, affiliates, and all those persons in active concert or participation therewith, from making, selling, offering for sale, using, and/or importing juvenile products, including the Deluxe Flutterbye Dreams Swing; the Flutterbye Dreams Swing; the Smart Stages 3 in 1 Rocker Swing; the Magical Mobile Swing; the Deluxe Quick Response Swing; the Deluxe Smart Response Swing; the Peaceful Time Open Top Swing; and the Fisher-Price Aquarium Take-Along Swing, or

otherwise directly or indirectly infringing the '113 and '624 Patents;

C. Award Graco damages adequate to compensate Graco for Fisher-Price's infringement, trebled pursuant to 35 U.S.C. § 284, together with interest and costs as fixed by the Court;

D. Enter an Order declaring this an exceptional case pursuant to 35 U.S.C. § 285 and award Graco its attorney fees; and

E. Grant Graco such further relief as this Court deems just under the circumstances.

Dated: October 27, 2005

DEMAND FOR JURY TRIAL

Graco demands trial by jury of all issues properly so triable under applicable law.

Lynn E. Rzonca

M. Kendall Brown

BALLARD SPAHR, ANDREWS & INGERSOLL,

LLP

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Philadelphia, PA 19103-7599

(215) 864-8109 (Telephone)

(215) 864-9278 (Telecopier)

Counsel for Plaintiff

GRACO CHILDREN'S PRODUCTS, INC.

EXHIBIT "A"

United States Patent [19]

Mitchell et al.

[54]	OPEN TOP SWING & CONTROL			
[75]	Inventors:	Daniel R. Mitchell, Morgantown; Scott B. Caley, Elverson; Truman Allison, York, all of Pa.		
[73]	Assignce:	Graco Childrens Products Inc., Elverson, Pa.		

[21] Appl. No.: 322,125

[22] Filed: Oct. 13, 1994

Related U.S. Application Data

[63]	Continuation-in-part of Scr. No. 13,747, Oct. 1, 1993.
[51]	Int. CL ⁶ A63G 9/16
[52]	U.S. Cl
[58]	Field of Search 472/118, 119;
	D6/347, 333, 344; 74/48; 185/40 C

[56] References Cited

U.S. PATENT DOCUMENTS

D. 250,861	1/1979	Boudreau et al
D. 293,046	12/1987	Richm .
D. 348,157	6/1994	Amburgey et al
D. 349,819	8/1994	Noll .
D. 351,289	10/1994	Stephens et al.,
1,282,927	10/1918	Paskal .
1,439,619	12/1922	Dziedzic .
1,458,049	6/1923	Grieshaber.
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1,906,768	5/1933	Romine .
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2,076,675	4/1937	Sharp .
2,564,547	8/1951	Schrougham .
2,609,031	9/1952	Puscas .
2,807,309	9/1957	Saint et al
2,908,917	10/1959	Pinson .
2,972,152	2/1961	Vincent .
2,979,734	4/1961	Saint et al
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3,128,076	4/1964	DiPasqua .
3,146,985	9/1964	Grudoski .
3,166,287	1/1965	DiPasqua .

1	111	Patent	Nin	nher
1	1111	A GLULIII		

5,525,113

[45] Date of Patent:

Jun. 11, 1996

3,459,423	8/1969	Meade .
3,526,400	9/1970	Carpenter et al.
3,692,305	9/1972	Allen.
3,818,517	6/1974	Casella .
3,842,450	10/1974	Pad.
4,150,820	4/1979	Bochmann .
4,452,446	6/1984	Saint .

(List continued on next page.)

FOREIGN PATENT DOCUMENTS

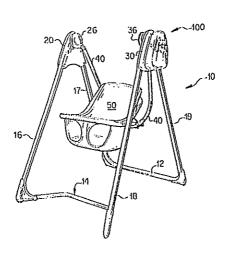
450755	8/1949	Italy .
497871	9/1954	Italy .
1070921	7/1967	United Kingdom.

Primary Examiner Kien T. Nguyen Attorney, Agent, or Firm Pennic & Edmonds

[57] ABSTRACT

An open top swing assembly uses a unique swing drive mechanism and a control to provide three selective swing height settings. The assembly has a frame which provides an open top structure for ease of access and a trapezoidal shaped front base to provide foot clearance. The swing drive mechanism includes a drive sleeve rotatably mounted to an axle that operatively supports the hanger. A drive flange is mounted on the axle, with a drive flange coupling device positioned between the sleeve and the drive flange to provide a limited lost motion connection. The coupling device includes a hub member coaxially and rotatably mounted on the axle and at least one torsional spring mounted coaxially on the hub member. The hub member includes abutments for engaging with the drive flange, whereby torque applied to the sleeve is transferred to the axle. A crank driven by a motor is linked to the sleeve to oscillate the sleeve. The swing height control device can have a sensor for detecting the swing height or amplitude. Preferably, three swing height settings are provided. The control device selectively outputs either no voltage, first, second or third predetermined voltages to selectively control the voltage input to the motor based on the selection of the swing height setting and/or the sensed swing height to achieve the selected swing height.

44 Claims, 13 Drawing Sheets



Case 2:05-cv-05696-LDD Document 1 Filed 10/27/05 Page 12 of 60

5,525,113 Page 2

Į	J.S. PAT	TENT DOCUMENTS	4,807,872	2/1989	Spilman et al	
		. ,	4,822,033	4/1989	Kohus et al	472/119
4,491,317			4,911,429	5/1990	Ogbu .	
1. 1.		Hyde et al McGugan et al	5,326,326	7/1994	Cunard et al	
		Casagrande .	5,326,327	7/1994	Stephens et al	

Jun. 11, 1996

Sheet 1 of 13

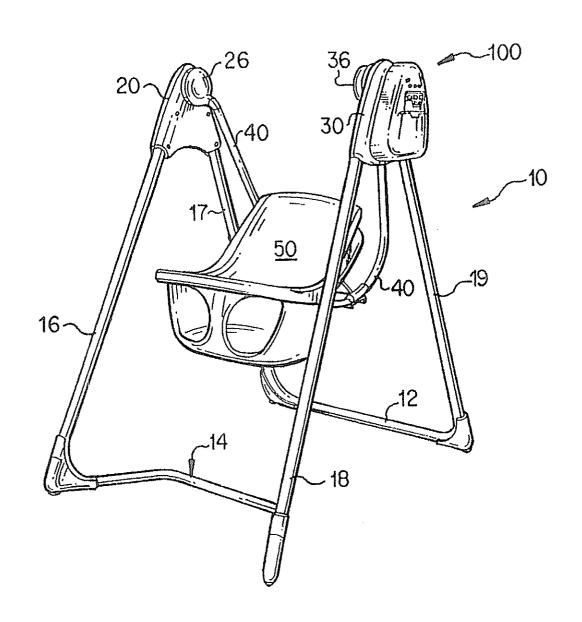


FIG. 1

5,525,113

1

OPEN TOP SWING & CONTROL

This is a continuation-in-part of design application Ser. No. 29/013,747 filed on Oct. 1, 1993.

BACKGROUND

Different types ok swings for an infant or child have been contemplated in the past. A swing typically comprises a support frame, a seat and at least one hanger attached to the 10 seat, the seat and the hanger defining a swing carriage, and a swing drive mechanism operatively connected to the hanger for maintaining the pendular movement of the swing carriage. If the swing carriage swings with no mechanical friction and no wind resistance, only a single push would be 15 needed to maintain the swing in a perpetual pendulum motion. In such a case, the swing will maintain its amplitude indefinitely and a swing drive mechanism would not be necessary. However, such is not the case in reality, as wind resistance and bearing friction are always present. The 20 mechanical or bearing friction can be reduced such that it becomes negligible. However, the wind resistance cannot be climinated. The bigger the child, the more wind resistance will there be. It is the wind resistance that mainly dampens the swing amplitude, requiring use of a swing drive mecha- 25 nism to supply energy lost and maintain its pendular move-

Typically, the swing drive mechanism is either electrically powered or manually powered. The electrically powered drive mechanism generally uses a DC or AC motor or solenoid, as described for instance in U.S. Pat. No. 4,452, 446 issued to Saint; U.S. Pat. No. 4,491,317 issued to Bansal; U.S. Pat. No. 4,722,521 to Hyde et al. The manually powered drive mechanism typically uses a spring wind-up mechanism which can be manually rotated using a crank to 35 store energy within the spring, as described for instance in U.S. Pat. Nos. 3,128,076 and 3,166,287 issued to Pasqua; and U.S. Pat. No. 3,459,423 issued to Meade.

SUMMARY

The present invention relates to an open top swing frame, an electrically powered swing drive mechanism, a swing height or amplitude control for providing selectable swing amplitudes, and an open top swing assembly using the same. 45 The open top support frame according to the present invention has a rear horizontal base, a substantially trapezoidal shaped front base, first, second, third and fourth legs, and first and second connectors. Specifically, the first and second legs extend upwardly, substantially parallel to one another, 50 at an incline from the ends of the rear base. Similarly, the third and fourth legs extend upwardly, substantially parallel to one another, at an incline from the ends of the front base. The first and third legs converge toward each other, as well as the second and fourth legs in a similar fashion. The first 55 and third leg pair and the second and fourth leg pair can be made substantially parallel and symmetrical to each other. A first connector is attached to the first and third leg pair to maintain them at a fixed position relative to each other and to the first connector. Similarly, the second connector is 60 attached to the second and fourth leg pair to maintain them at a fixed position relative to each other and to the second

The rear and front bases are substantially on the same plane, namely on the floor to support the entire frame 65 thereon. The trapezoidal shaped front base has its median arm joined by a pair of laterally and forwardly extending

2

arms so that the opening thereof faces away from the rear base, or rather faces toward the front. The median arm is substantially parallel to and closer to the rear base. The opening created by the trapezoidal shaped front base provides an obstruction free foot clearance for the person seating or removing an infant or child from the swing.

A first pivot or pendulum axle is rotatably journaled to the first connector and a second pivot or pendulum axle is rotatably journaled to the second connector. A pair of hangers extending laterally from the seat can be connected to the first and second pendulum axles such that the seat can oscillate thereabout. Preferably, the first and second axles are aligned so that their axes are collinear about a same horizontal axis.

While it is not necessary, a hub can be use to connect the axles to the hangers, with one of the hangers mounted to one of the hubs and the other of the hangers mounted to the other of the hubs. Each of the hubs can have an overrotation stop which cooperates with a cooperating overrotation stop mounted on each of the first and second connectors adjacent to each of the hubs to prevent overrotation of the hubs relative to the first and second connectors and thus prevent overrotation of the swing carriage.

Another feature of the present invention is a swing drive mechanism. Although it is preferable to use an open top swing frame described above with the drive mechanism according to the present invention, the present drive mechanism can be used with any conventional swing. The drive mechanism comprises a drive sleeve mounted coaxially and rotatably about an axle so that it can substantially freely rotate thereabout. A drive flange is mounted on the axle with no relative rotational movement therebetween. A drive flange coupling device is positioned between the drive sleeve and the drive flange to cause the axle to oscillate with the sleeve in the same direction. A crank driven by a motor via a gear reduction train is linked to the sleeve to oscillate the sleeve and thus the axle via the coupling device and the drive flange.

The sleeve includes a channel radially spaced from the axle and extends parallel with the axle. The crank basically rotates about an axis that is perpendicular to the axle. The crank has a driven portion that is offset from the axis of rotation of the crank. Accordingly, rotation of the crank causes its offset driven portion to follow a circular orbit path whose radius is the distance of the offset. The offset driven portion preferably has a ball that is rotatably mounted thereabout. The ball is slideably mounted in the channel such that rotation of the crank enables the sleeve to oscillate about the axis of the axle while the ball slideably oscillates back and forth within the channel. Means other than the ball, such as a cylinder or universal pivot, can be attached to the driven portion to carry out the same function.

The coupling device comprises a hub member coaxially and rotatably mounted on the axle and at least one torsional spring mounted coaxially on the hub member. The hub member includes abutments for engaging with the drive flange, whereby torque applied to the sleeve is transferred to the spring which can cause the hub member to rotate relative to the axle which in turn can cause the abutments to engage the drive flange and transfer torque to the axle. Preferably, the spring is provided with a limited free play and sufficient travel before it engages with the sleeve and to allow the swing carriage to swing when the motor is stopped, or to allow the motor to rotate when the swing carriage is stopped, without causing damage to the swing drive mechanism. During the interim when the free play (lost motion) is

operational, the sleeve is decoupled from the axle and thus from the swing carriage.

The motor has its output shaft mounted substantially perpendicularly to the axle with the crank rotating about an axis perpendicular to both the output shaft and the axle. 5 Preferably, a flywheel is attached to the motor.

Another aspect of the present invention is a swing height or amplitude control which can be used with the swing drive mechanism according to the present invention. The swing height control according to the present invention, however, 10 can be used to control any conventional swing having a motor operated swing drive mechanism. The control can provide at least two swing height settings (first and second), where the first setting is smaller than the second setting, where it simply outputs either a first or second predeter- 15 3, showing the details of the swing drive mechanism. mined voltage to the motor based on the selection of the swing height setting, where the first voltage is lower than the second voltage.

The control can also include a sensor for continuously detecting the swing height or amplitude. Where the control 20 provides at least first and second swing height settings, the control can output either no voltage, a first predetermined voltage or a second predetermined voltage to selectively control the voltage input to the motor based on the selection of the swing height setting and the sensed swing height to 25 achieve the desired swing height. The control can also provide three or more swing height settings (first, second, and third), with the third setting being the largest. In this regard, the control selectively outputs either no voltage, the first predetermined voltage, the second predetermined volt- 30 age or a third predetermined voltage, with the third being the greatest. The control can be made to output as many (or more) different voltage outputs as there are different swing amplitude settings.

In operation, using the sensor with the three height 35 setting, upon selection of the first swing height setting, the first voltage is continuously applied to the motor regardless of the swing height detected. Preferably, when and if the detected swing height exceeds the selected swing height setting, the voltage can be cut-off to the motor for the 40 duration of the portion of the swing cycle that exceeds the selected first height setting to provide a more accurate swing height setting.

If the second swing height setting is selected, again the first voltage is initially input to the motor until the detected swing height exceeds the first swing height setting. Upon the swing height exceeding the first swing height setting, the second voltage is applied to the motor only for the duration of the portion of the swing cycle that exceeds the first swing 50 height setting.

If the third swing height setting is selected, again the first voltage is initially applied to the motor until the detected swing height exceeds the first swing height setting. Upon the swing height exceeding the first swing height setting, the 55 third voltage is applied to the motor for the duration of the portion of the swing cycle that exceeds the first swing height setting.

Preferably, when and if the swing height is greater than the third swing height setting, to prevent excessively high 60 swing height, the first voltage is applied to the motor for the duration of the portion of the swing cycle that exceeds the third swing height setting.

BRIEF DESCRIPTION OF THE DRAWINGS

These and other features, aspects, and advantages of the present invention will become much more apparent from the

following description, appended claims, and accompanying drawings where:

- FIG. 1 is a perspective view of an open top swing according to the present invention.
- FIG. 1A is a top elevational view of a portion of FIG. 1, showing the front base of the open top swing frame according to the present invention.
- FIG. 2 is an enlarged side view of the right leg connector which houses the swing drive mechanism and associated
- FIG. 3 is a perspective view of FIG. 2, with its cover removed, showing the swing drive mechanism.
- FIG. 4 is a sectional view taken along line 4-4 of FIG.
 - FIG. 5 shows the details of the motor and the crank.
- FIG. 6 is a sectional view of the right connector with the hub, showing the overrotation stops formed on the connector and the corresponding overrotation stop formed on the hub for limiting the swing amplitude of the swing carriage.
- FIG. 6A is a perspective view of the left leg connector with its hub removed therefrom to show its pendulum axle and its overrotation stops for limiting the swing amplitude of the swing carriage.
- FIGS. 7A,7B, 8A,8B, 9A,9B and 10A,10B show the operation of the swing drive mechanism and the relative position of the crank relative to the sleeve member.
- FIG. 11 is an exploded view of the drive mechanism arrangement, including the sleeve, the flange drive coupling device, the drive flange and the axle.
- FIG. 12 is an exploded view of the drive flange and the drive coupling device arrangement, taken along line 12-12 of FIG. 11.
- FIG. 13 is sectional view taken along line 13—13 of FIG. 4, showing the drive flange and a swing position detector.
- FIG. 14 is a schematic elevational bottom view of the prongs taken along line 14—14 of FIG. 11.
 - FIG. 15 is a schematic representative of a pendulum.
- FIG. 16 shows one embodiment of the controls for the swing drive mechanism.
- FIG. 17 shows another embodiment of the controls for the swing drive mechanism.

DESCRIPTION OF THE DRAWINGS

FIG. 1 shows an embodiment of a swing according to the present invention, which has a support frame 10 which holds a swing drive mechanism 100, a pair of hangers 40, and a seat 50. The support frame 10 according to the present invention has an open top design. It has no overhang support member to make removal and seating of an infant to and from the swing seat convenient. The open top frame 10 has a rear horizontal base 12, a substantially trapezoidal shaped front base 14, a front left leg 16, a rear left leg 17, a front right leg 18 and a rear right leg 19 in a splayed position as shown in FIG. 1, a left leg connector 20 and a right leg connector 30. The rear left and right legs 17,19 extend upwardly, substantially parallel to one another, at an incline or angle from the ends of the rear base 12. Similarly, the front left and right legs 16 and 18 extend upwardly, substantially parallel to one another, at an incline from the ends of the front base 14. The front and rear left legs 16,17 incline in the opposite directions such that they converge toward each other as shown in FIG. 1. Similarly, the front and rear right legs 18,19 incline in the opposite directions such that

they too converge toward each other. The front and rear left leg pair 16,17 can be substantially parallel and symmetrical to the front and rear right leg pair 18,19 if desired.

The left leg connector 20 connects the front and rear left legs 16 and 17 to maintain them at a fixed position relative 5 to each other. Similarly, the right leg connector 30 connects the front and rear right legs 18,19 to maintain them at a fixed position relative to each other.

The rear and front bases are substantially on the same plane, namely on the floor to support the entire frame thereon. The front base is substantially trapezoidal shaped. Specifically, as shown in FIG. 1A, the front base is formed by a horizontal median arm 14a joined by a pair of oppositely extending arms 14b,14c. The arms 14b,14c are angled greater than 90° with respect to the median arm 14a such that they form a trapezoidal shape. The front base extends inwardly toward the rear base with the median arm 14a preferably parallel to the rear base. Due to this feature, the front base provides an opening or clearance space which enables one to move close to the seat during seating or removal of an infant or child from the swing, i.e., foot clearance.

As shown in FIGS. 3, 4, 6, and 11, a right pendulum axle 32 is rotatably journaled via axially spaced apart bearings 34 or the like on the right leg connector which houses the swing drive mechanism 100. A left pendulum axle 22 can be rotatably journaled via axially spaced apart bearings 24 or the like on the left leg connector in the similar fashion. The ends of the left and right hangers 40 which extend laterally from the seat 50 can be operatively connected to the left and right pendulum axles, respectively, to enable the seat to swing or oscillate about the axles. The left and right pendulum axles can be aligned so that their axes are collinear about a same horizontal axis to maintain an equal pendulum left and right hanger length.

According to the present invention, left and right hubs 26,36 are preferably connected to the left and right pendulum axles, respectively, with no relative rotational movement between the hubs and their axles. The left hanger is 40 mounted to the left hub 26 and the right hanger to the right hub 36. As shown in FIGS, 4, 6 and 6 Λ , each of the hubs preferably has means cooperating with their respective left and right leg connectors 20,30 for limiting the degree of rotation. Specifically, the limiting means comprises at least 45 one overrotation stop 60, a pair of stops being preferable as shown in FIGS. 6 and 6A, extending laterally from each leg connector 20,30. The stops 60 cooperate with cooperating overrotation abutments or stops 62 formed on each of the hub 26,36 to prevent overrotation of the hubs relative to the 50 connectors and thus the swing carriage. The maximum degree of rotation Θ_{MAX} between the abutments is about 70° or the swing amplitude of about 35° as schematically shown in FIG. 6.

A swing can generally be considered to behave as a simple 55 pendulum when the amplitude is relatively small, where the period of oscillation is also generally unaffected by the mass of the pendulum. The swing amplitude is preferably between about 0° to 22° as presently contemplated by an embodiment of the present invention, which means that the period of 60 oscillation for the swing is more or less can be considered to be substantially constant between these amplitudes. The velocity of the pendulum is greatest at its neutral position, i.e., swing amplitude of 0° and smallest at its peak amplitude (zero velocity) where it changes its direction. When the 65 period is constant, a pendulum swinging at a bigger amplitude will have to travel at a greater velocity than the same

swinging at a smaller amplitude. That is, a pendulum swinging at a bigger amplitude has to travel further during the same period and thus has to travel faster. In this regard, the drive mechanism needs to accommodate not only for variations of speed of the swing carriage, it must be synchronized with the swing cycle in order to achieve a natural swing

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The present invention contemplates a novel swing drive mechanism which operates in synchronism with the swing cycle regardless of the swing amplitude. Preferably, the present swing drive mechanism can selectively maintain two or more different levels of swing amplitude or swing speed, i.e., low, medium and high, for example. The swing drive mechanism 100 according to the present invention is shown in PIGS, 3-5 and 7-12. Although it is preferable to use an open top swing frame described above with the swing drive mechanism according to the present invention, any conventional swing frame can be used. The swing drive mechanism 100 comprises a drive sleeve 110 mounted coaxially and freely rotatably about the axle 32, a drive flange 120 is mounted substantially collinearly adjacent the drive sleeve on the axle with no relative rotational movement between the axle and the drive flange. A drive flange coupling device 130 is positioned between the drive sleeve and the drive flange, and a crank 150 driven by a motor 160 via a gear reduction train 155,156 is linked to the sleeve to oscillate the sleeve and thus the axle 32 via the coupling device and the drive flange.

As better shown in FIGS. 11 and 12, the drive flange 120 comprises a disc member 121 with a central circular flange 122 extending collinearly therewith from the inner side or face 123 thereof. A central hole 124 extends through the flange and the disc member, which is provided with conventional means for limiting the rotational movement of the disc member relative to the axle, such as a non-circular hole, i.e., a square-shaped, D-shaped, V-shaped or crescent-shaped openings, etc., as shown in FIG. 11, which cooperates with a complementary shaped axle. The inner side 123 of the disc member is provided with a recess 125 having five symmetrical divisions, substantially akin to a propeller or five-leafed clover. Each of the five divisions has opposed abutment side walls 125a,125b.

The disc member 121 also has a radial extension 126 extending radially therefrom. An abutment 128 extends substantially perpendicularly from the free end of the extension 126. The abutment 128 also extends coaxially and circumferentially about the axle 32, parallel with the axle, and has two opposed abutment edges 128a,128b formed by the parallel edges thereof.

The coupling device 130 comprises a hub member 140 coaxially and rotatably mounted on the axle and at least one torsional spring 133,134 mounted coaxially on the hub member. Although the drawings show two discrete springs, a single continuous torsional spring attached to the hub member can also be used. The hub member has a central throughhole 141 slightly larger than the outer diameter of the flange 122 so that it coaxially engages thereover and freely rotates thereabout. The hub member preferably has a pentagonal central flange 142 collinearly arranged about a star-shaped disc 143 which has five symmetrical radial extensions 143a. Any non circular central flange can be used so long as it does not permit the spring to rotate thereabout. Each of the extensions 143a is substantially narrower than the distance between the abutment walls 125a,12b formed on each of the five divisions of the recess to enable the hub member to freely rotate relative to the drive flange 120, for example, of about 20°.

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As shown in FIG. 11, two discrete torsional springs 133,134 of substantially equal spring constant are preferably positioned between the sleeve and the hub member and coaxially wrapped around the hub member in the opposite directions with no relative rotational movement between the hub member and the springs. Each of the springs has a substantially pentagonal central opening which corresponds to the pentagonal flange 142 of the hub member to enable the springs to be mounted coaxially thereon with no relative rotational movement. Each of the spring has a hook 135,136 facing toward each other for engaging with the sleeve. As previously indicated, a single spring attached to the hub member, for instance by way of a slot, with their ends capable of engaging the sleeve can also be used rather than two springs if desired.

The sleeve 110 comprises a substantially cylindrically shaped body 111 collinearly formed with a tear drop shaped plate member 116 having a planar outer face 116a, with a central throughhole 112 extending through the cylindrical body and the plate member. The throughhole 112 is dimensioned to enable the sleeve to freely rotate about the axle 32. The body 111 is preferably provided with a plurality of radially extending reinforcement ribs 113 and a channel 114 radially spaced from the axle and extending parallel with the cylindrical body.

The drive sleeve engages the springs via a spring engaging element 115 extending axially from the apex of the tear drop shaped plate member. The engaging element is axially and angularly aligned with the channel. The spring engaging element is also formed radially further away from the 30 throughhole than the channel and can be aligned with the abutment 128. Two opposed abutment edges 115a and 115b are formed by the lateral edges of the spring engaging element 115. The distance between the abutment edges 115a,115b is preferably about same as that between the $_{35}$ abutment edges 128a,128b, but smaller than the distance between the two hooks 135,136 such that the sleeve can freely move relative to the springs for a limited degree (providing a free play or lost motion relationship), which in turn translates to lost motion or free play relative to the axle 40 32. Specifically, unless the spring is already adjacent to one of the abutment portions 115a,115b, the sleeve has to rotate relative to the spring before it engages one of the springs and cause the hub member 140 to rotate and abut the drive flange

The springs are arranged such that they engage opposed abutment edges of the abutments 115,128 and tend to cause the springs to coil tighter around the pentagonal central flange 142. Specifically, the two springs are coiled in the opposite directions such that rotation of the sleeve 110 in the clockwise direction (CW) causes the abutting edge 115a thereof to engage the hook 135 while causing the abutting edge 128b to engage the hook 136. Rotation of the sleeve in the counterclockwise direction (CCW) causes the abutting edge 115b thereof to engage the hook 136 while causing the abutting edge 128a to engage the hook 135.

The load required to oscillate the swing carriage at a relatively low amplitude, for instance of 10°, is generally relatively small. However, the energy required to oscillate increases by the square as the amplitude increases. In order 60 to accommodate for varying loads, the present invention contemplates use of a spring or springs, in conjunction with the free play arrangement, to provide a plurality of spring gradients, three to be specific, to accommodate different swing heights. Specifically, the free play arrangement 65 (where the relative differences between the width of the abutment 128 and the distance between the hooks 135,136)

enables the sleeve to rotate freely relative to the spring. The free play provides the first gradient of zero load for a first predetermined angle of rotation. When the sleeve is rotated relative to the axle beyond the first predetermined angle of rotation in the same direction, one of the abutment edges 128a,128b is engaged with one of the hooks 135,136 and the other of the hooks 135,136 is engaged with one of the abutment edges 115a,115b, both springs being engaged such that they both become active. When the two springs are active, they provide a second gradient of load for a second predetermined angle of rotation. The second predetermined angle of rotation is preferably small relative to the first angle of rotation, which can begin when the load necessary to increase the swing amplitude increases relatively sharply to preferably parallel the load requirement for the corresponding swing amplitude. When the sleeve is rotated beyond the second predetermined angle of rotation in the same direction, the radial extensions 143a abut against one side of the side walls 125a,125b, preventing the hub member from rotating relative to the drive flange. When this happens, only one of the springs, the spring engaging the sleeve, becomes functional, which provides the third spring gradient which is substantially greater than the second spring gradient again to parallel the load requirement for a greater swing amplitude. In essence, if the spring constant between the two springs is equal, the third spring gradient would increase about two folds since only one of the two opposingly acting springs becomes active. These three spring gradients can provide the necessary load constants to operate a swing having variable swing amplitudes.

Referring to FIG. 4, the swing drive mechanism is housed in the right leg connector 30, but can just as easily be housed in the left leg connector 20. The axle 32 is rotatably journaled to the connector 30 to enable the axle to pivot or oscillate to cause the hub 36 to rotate along with the axle to thereby oscillate the hanger connected thereto. According to the present invention, the sleeve is caused to oscillate using a crank 150 which is driven preferably by a DC motor 160. As previously indicated, it is desirable to prevent the motor from straining or seizing when the seat is stopped from swinging, intentionally or otherwise while the motor is running. The torsional springs 133,134 in conjunction with lost motion arrangement (of the sleeve relative to the axle) can absorb the energy input by the motor in the event the swing carriage is stopped while the motor is running or in the event the motor is stopped while the swing carriage is in motion. During the interim when the lost motion is operational, the sleeve is basically decoupled from the axle and thus from the swing carriage. In this regard, the free play or the lost motion arrangement can enable the axle to oscillate less than the amplitude driven by the crank, as will be explained from below.

The crank 150 basically rotates about an axis 151 that is perpendicular to the axle. The crank has a driven portion 152 that is offset from and parallel to the axis 151 of rotation of the crank. Rotation of the crank thus causes its offset driven portion to follow a circular orbit whose radius R is the distance of the offset. In this regard, the radius of the offset should be such that the orbiting crank oscillates the sleeve at a greater amplitude than the greatest desired oscillation (third amplitude).

The offset drive portion 152 preferably has a ball 153 that is rotatable about the driven portion, the ball being slideably mounted in the channel such that rotation of the crank enables the sleeve to oscillate about the axle while the ball slideably oscillates back and forth within the channel. To properly track the ball within the channel, the length of the

channel should be same or longer than the diameter of the orbiting ball. Means other than the ball, such as a cylinder, universal pivot or flexible link can be attached to the driven portion to enable transfer of orbiting motion to oscillatory motion.

As shown in FIG. 5, the crank is fixedly connected to a drive train which includes a driving gear 155 engaged to a worm shaft 156 which is connected to an output shaft 162 of the motor 160. The output shaft 162 is mounted substantially perpendicular to the axle, and the crank rotates about the 10 axis 151 that is perpendicular to the output shaft 162 and the axle 32. Preferably, the motor has a flywheel 164 connected to the output shaft 162 to even the varying load (encountered during the swing cycle) applied to the motor. The motor and the crank are preferably housed in a motor housing 170 which is non-displaceably connected to the connector 30. The crank and the motor rotate about their axes of rotation which does not change relative to each other, to the axle 32 or to the connector 30.

FIGS. 7–10 show the schematic position of the crank in relationship to the sleeve. FIGS. 7A, 8A, 9A and 10A are views taken along the line A—A of FIG. 3, with the drive flange 120, the hub member 140 and the springs 133,134 omitted for convenience of illustration. FIGS. 7B, 8B, 9B and 10B are views similar to FIG. 4, but showing only the motor housing 170, including the motor 160 and the crank 150, and a section of the channel 114 formed on the sleeve 110. As seen from arrows W, the crank rotates in one direction.

FIGS. 7A and 7B show the instance where the sleeve has rotated counter-clockwise (CCW) and reached its maximum amplitude Θ_1 , Θ_2 , or Θ_3 as shown in FIG. 15. At this instance, the force vector V output by the crank is substantially parallel to the axis of rotation of the sleeve, thus imparting no oscillatory motion. The sleeve is moving at zero velocity and changing its direction of rotation. As seen from FIG. 7B, the offset driven portion 152 is positioned about the midpoint of the channel, with the ball 153 slid up relative thereto as shown by the arrow U.

FIGS. 8A and 8B show the instance where the crank has rotated 90° relative to the crank positioned in FIGS. 7A and 7B, respectively, causing the sleeve to rotate in the opposite direction. At this instance, the sleeve is rotating in the clockwise (CW) direction at its maximum velocity, with the ball slid down as shown by arrow D to its lowest point relative to the offset driven portion. At this instance, the force vector V output by the crank is perpendicular to the axis of rotation of the sleeve, where the velocity of the rotating sleeve is substantially equal to the orbiting velocity of the crank. As shown in FIG. 8B, the offset driven portion is at its rightmost point on the channel.

FIGS. 9A and 9B show the instance where the crank has rotated about 90° relative to the crank positioned in FIGS. 8A and 8B, respectively. In this instance, the sleeve has rotated clockwise (CW) and reached its maximum amplitude Θ_1 , Θ_2 , or Θ_3 . Again, the force vector V output by the crank is parallel to the axis of rotation of the sleeve at this point. Thus, the sleeve is moving at zero velocity and changing its direction of rotation. As seen from FIG. 9B, the offset driven portion is positioned about the midpoint of the channel, with the ball slid up relative thereto as shown by the arrow U.

FIGS. 10A and 10B show the instance where the crank has rotated about 90° relative to the crank positioned in FIGS. 9A and 9B, respectively, causing the sleeve to rotate 65 in the opposite direction. At this instance, the sleeve is rotating in the clockwise (CCW) direction at its maximum

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velocity, with the ball moved down as shown by the arrow D to its lowest point relative to the offset driven portion. Again, the force vector V output by the crank is perpendicular to the axis of rotation of the sleeve, where the velocity of the rotating sleeve is substantially equal to the orbiting velocity of the crank. As shown in FIG. 10B, the offset driven portion is at its leftmost point on the channel.

It was already described that the velocity of the pendulum is greatest at its neutral position, i.e., swing amplitude of 0° and zero at its peak amplitude where it changes its direction. The sleeve/crank arrangement according to the present invention substantially mimies the pendulum motion, where the velocity of the oscillating sleeve is greatest where its amplitude is at 0° and zero at its maximum amplitude where the direction of rotation changes.

The drive mechanism according to the present invention accommodates not only for variations of speed of the swing carriage to achieve a natural swing motion. This is achieved by using the above described crank/sleeve arrangement in conjunction with the above described drive flange coupling device 130 which has three different spring gradients or constants. Specifically, the oscillation amplitude of the sleeve will remain substantially constant at Θ_S as schematically represented in FIG. 15, generally limited by the orbit diameter of the driven portion. However, due to the lost motion or free play arrangement described above in conjunction with the springs, the axle does not need to oscillate the same amount. Depending on the amount of torque output by the motor, the axle can always be controllably driven less than the oscillation amplitude of the sleeve.

Specifically, the crank can be tuned to oscillate the sleeve at a period substantially equal to the natural oscillation period of the swing carriage to synchronize the sleeve with the oscillation of the swing carriage. With reference to FIG. 15, if the torque applied to the motor is such that the swing carriage can only oscillate a fraction of the oscillation amplitude, at Θ_1 for instance, the lost motion arrangement can enable the sleeve to oscillate to Θ_S . Since the period of oscillation is the same for the sleeve and the swing carriage, the sleeve will remain synchronized with the swing carriage. Any small synchronizing discrepancy occurring between the sleeve and the swing carriage due to mechanical aberration can be absorbed by the loss motion arrangement and the springs to maintain proper synchronization.

The swing mechanism described above can be used with any conventional swing control. For instance, to provide two different amplitudes, low and high, one can provide a control that outputs two different voltages depending on the swing height selected. Upon selection of the low amplitude setting, a low voltage can be input to the motor. Upon selection of the high amplitude setting, a relatively higher voltage can be input to the motor. Preferably, the motor operates substantially at a constant speed regardless of the voltage input to the motor. By inputting higher voltage, the motor will impart a greater torque to cause the axle to oscillate at a relatively greater amplitude.

Another aspect according to the present invention is a unique swing height or amplitude control 200 which can be used with the swing drive mechanism described above or with any conventional swing. According to the present invention, the swing control incorporates means for detecting the swing height or amplitude, which can be any conventional switches which can be triggered by any element that oscillates with the seat such as the hanger or the pendulum axle.

The swing control 200 according to the present invention, can provide three swing height or speed settings (first,

second, and third), where the first setting is smallest, the third setting the largest and the second setting falling between the first and second settings. The swing control can selectively output either zero voltage, a first predetermined voltage, a second predetermined voltage or a third predetermined voltage to selectively control the voltage input to the motor based on the swing height or speed selected and the sensed swing height to achieve the desired swing height. The first, second and third voltages are greater than zero, with the first voltage being the smallest and the third being the greatest, with the intermediary second voltage falling between first and second voltages.

According to the present invention, the swing height detection means shown in the preferred embodiment comprises a swing angle indicator formed on the drive flange 120 and a light interrupt detector 210. As shown in FiGS. 11 and 14, the angle indicator comprises a pair of spaced apart prongs 127 extending substantially perpendicularly from the free end of the extension 126. The prongs 127 extend coaxially and circumferentially about the axle 32, parallel with the axle, in the direction opposite the abutment 128. The dimensions of the two prongs are substantially the same, with the spacing between the prongs being about the width of one of the prongs. The prongs operate in conjunction with a light interrupt indicator 210 to determine the angle of rotation of the axle relative to the connector.

The light interrupt detector 210 comprises a photodetector or phototransistor 212 aligned with and spaced apart from an infrared light emitting diode (IRLED) 214. Since the drive flange is non-rotatably connected to the axle, the prongs 30 rotate along with the axle 32. As shown in FIGS, 4 and 13, the light interrupt detector is positioned so that the prongs can oscillate between the photodetector and the IRLED. As the prongs oscillate, they can interrupt or block light emitting from the IRLED to the photodetector, representative of the swing amplitude exceeding a predetermined setting. The prongs and spacing therebetween are dimensioned such that they can indicate at least three different patterns of light interruption to detect the swing amplitude. Specifically, when the oscillation occurs between the prongs (within the 40 spacing between the prongs), light emitting from the IRLED is not interrupted. In this mode, the swing height is within the first swing height setting. When the oscillation is greater such that the prongs do interrupt light emitting from the IRLED, the swing is oscillating within the second or third swing height setting. When the oscillation occurs even at a greater angle, the prongs interrupt light emitting from the IRLED as in the second swing height setting, but the prongs can swing past its extreme outer edges 127a,127b, which at that point ends the light interruption (within the same 50 period). In this mode, the swing oscillates past the third swing height setting.

Depending on the amplitude of the swing, the prongs either interrupt or do not interrupt light emitted by the IRLED. When the swing is centered (at its neutral position), 55 the amplitude Θ is at 0° as shown by schematic representations in FIGS. 14 and 15. The prongs can be dimensioned, for instance, so that the amplitude at Θ_1 is about 9° and at Θ_3 is about 22° . The prongs do not interrupt light emitted by the IRLED until the prongs rotate either direction from the center by an amplitude of about 9° . From the amplitude of about 9° to the amplitude of about 22° , the prongs interrupt light emitting from the IRLED. When the prongs rotate beyond about 22° amplitude, the light becomes uninterrupted.

FIGS. 16 and 17 show schematic representative block diagrams of different embodiments of the control according

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to the present invention which can selectively produce a plurality of different voltages which can be applied to the motor in order to produce three different swing amplitudes. For convenience, the same or equivalent elements have been identified with the same reference numerals. The amplitudes are referred to as low (first), medium (second) and high (third), which are actuated by switches, preferably pushbuttons 301, 302, and 303, respectively. A stop switch, preferably pushbutton 304 is provided for turning off the control.

According to the preferred embodiment, a switch interface 300 is provided between the switches LOW 301, MED 302, HIGH 303 and STOP 304 and their respective LOW LED 305, MED LED 306 and HIGH LED 307. The interface can include a conventional circuitry which remembers the last switch depressed, such as a non-clocked flip-flop(s). The control can include a power on switch. However, since such an interface typically uses an insignificant amount of power, it can remained powered to eliminate the need for a separate power on switch. When any one of the switches 301, 302 and 302 is turned on, a digital RUN output signal 309 and the corresponding "L", "M", or "H" digital signal become high. These digital signals then control other control elements. Specifically, the run output enables power to be supplied to the control elements or circuitry 320, 330, 330', 360 (360') and 380, 380'. When the STOP switch is pushed, the RUN output becomes disabled or turned low to shut off the control. Any switch can be activated at any time regardless of the previous selection.

The "H" and "M" outputs can be connected to two opposite outputs of a flip-flop to make them complements of each other. Accordingly, whenever "H" output from the switch interface box 300 becomes high, the "M" output will be low and vice-versa. When either the HIGH or MED switch is activated, the "L" output becomes low, for instance, by grounding the "L" output signal, to cause the LED bias current to flow through resistor RLIMIT 321 and light the HIGH or MED LED. On the other hand, whenever the LOW switch is activated, the "L" output becomes high, regardless whether the "H" or "M" output is high and the diode 308 in series with the MED or HIGH LED will cause all the biasing current to be shunted through the LOW LED 307. The "H" or "M" output can remain high as this signal will have no effect on the voltage output when the LOW switch is turned on.

Whenever any one of the LOW, MED and HIGH switch is activated, the RUN output becomes high and a predetermined reference voltage (PRV) can be generated by the reference generator 330 and applied to the voltage regulator 380 as shown in the embodiment of FIG. 16. Alternatively, the voltage regulator can produce its own reference voltage as shown in the embodiment of FIG. 17. In addition, the reference generator 330 and the IRLED bias circuit 330 produce the necessary bias voltage for the IRLED 214.

When the RUN output is high, the pulse-width modulation (PWM) circuitry or switch 384 becomes active. A resistor divider network comprising RF 381 and RLOW 382 can provide a percentage of output voltage as a feedback value to the voltage regulator 380. Whenever the feedback value is less than the PRV, the PWM switch 384 is closed. This causes the averaging capacitor 383 voltage to rise until the feedback voltage value becomes greater than the PRV (plus a small amount of hysteresis). At this point, the PWM switch 384 opens and the capacitor voltage decays. When the capacitor voltage decays down to the PRV (minus a small amount of hysteresis), the PWM switch 384 is closed once again, repeating the process to maintain the average value of the feedback voltage to equal the PRV. The output voltage to

the motor is controlled by resistors RLOW 382, RMEDIUM 386, RHIGH 387 and RP 381 since the feedback voltage represents a fixed percentage of the output.

It is important to note that because the motor acts as an inductive load, when the PWM switch is opened, current 5 still flows into the motor. The flyback diode 385 can be used to provide a path for this current and clamp the output voltage to a diode voltage, typically 0.5 to 0.7 V below ground.

With respect to the embodiment shown in FIG. 16, when to the LOW switch 301 is activated, the "L" signal becomes high and the AND gate 362 will always output a low signal. Accordingly, the IR switch 361 will always be held open by the AND gate 362 when the LOW switch is activated, and the feedback percentage, as described above, can be defined solely by the RLOW 382 and the RF 381. This is true for the low swing amplitude setting regardless of the position of the prong 127 or the values of the "M" or "II" signal output to the output voltage level switching circuit 360. Alternatively, as shown in FIG. 17, the output voltage level switching 360' can cut-off the voltage upon the prongs 127 interrupting light emitting from the IRLED. In this embodiment, an OR gate 365 can be used to selectively provide high or low ENABLE signal to the voltage regulator 380'. Specifically, when the "L" signal is high and the prongs do not block light emitting 25 from the IRLED, the OR gate will always produce high ENABLE signal. However, when the prongs do block light emitting from the IRLED, the OR gate will produce a low ENABLE signal to disable the voltage regulator, providing no voltage output to the motor. The values of the RLOW and the RF thus can be selected to provide the desired low or first output voltage to the motor.

When the MEDIUM switch 302 is activated, the "M" output goes high, the "M" switch 363 is closed and the "L" input to the inverter of the AND gate 362 becomes low. The 35 IR switch 361 is closed only when the photodetector 212 outputs high signal, i.e., interruption of light emitting from the IRLED. Accordingly, the voltage output to the motor will be controlled by the RLOW 382 and the RF 381 as in the LOW mode. Alternatively, with respect to FIG. 17, the 40 low "L" signal is output to the inverters of the OR gate 365 and the AND gate 366, while either high or low signal from the photodetector 212 is input to the AND gate 366 and the inverter of the OR gate 365. Since the "L" signal will always be low in this mode, the OR gate will always output a high 45 ENABLE signal and always enable the voltage regulator. Again, the IR switch will close only when the photodetector outputs high signal (upon interruption of the light). When the IR switch 361 is closed, the RMEDIUM 386 is connected in parallel with the RLOW 382, lowering the overall 50 resistor value and thus the feedback percentage to raise the voltage output to the motor to the selected medium voltage level. The value of the RMEDIUM thus can be selected to provide the desired medium or second output voltage to the molor.

The operation of the HIGH mode is substantially similar as the MEDIUM mode. Specifically, when the HIGH switch 303 is activated, the "H" signal goes high and the "L" signal goes low. The IR switch 361 is closed only when the photodetector outputs high signal upon the prongs interrupting the light. However, when the IR switch 361 is closed, the RHIGH 387 is connected in parallel with the RLOW 382. The value of the RHIGH can be selected to provide the desired high or third output voltage to the motor. Alternatively, the "M" switch 363 can be closed along with the 65 "high" switch to connect the RHIGH, the RMHDIUM and the RLOW in parallel. In this regard, a higher value RHIGH

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can be used to provide the same high or third output voltage to the motor,

In the medium and high swing settings, whenever the IRLED light is not interrupted such that the photodetector outputs a high signal, the IR switch 361 is opened or remains opened, preventing both the RMEDIUM and the RHIGH from being connected in parallel with the RLOW. This forces the output voltage to its low value regardless of the position of the "M" switch 363 or the "H" switch 364.

The PRV generated by the reference generator 330 can be produced for example by a semiconductor diode, which typically has a negative temperature coefficient of about -2 millivolt/°C. Thus, as the temperature increases, the reference output voltage from the semiconductor diode falls. Accordingly, when such a semiconductor diode is used, it is desirable to provide a temperature compensator 370 such as a negative coefficient thermistor connected in parallel with the RLOW 382 to compensate for the drop in reference voltage. As the temperature increases, the thermistor resistance decreases, thereby decreasing the percentage of feedback. This action increases the output voltage with increasing temperature and thus compensates for the fall of the reference voltage. Alternatively, as shown in FIG. 17, a temperature compensator can be built into the voltage regulator 380' which produces its own internally temperature compensated reference voltage VOLT REE.

Preferably, the battery sense and flash circuit 320 can be used to cause at least one of the indicator LEDs 305, 306, or 307 to flash when the battery voltage supply falls below a predetermined voltage level to provide a visual indication of when the batteries need to be replaced.

As described above, the exemplary controls shown in FIGS. 16 and 17 can be used to produce three different output voltages to the motor depending upon the swing amplitude selected by the user. However, modifications can be made to the control shown in FIGS. 16 and 17 to achieve the same functional attributes. For example, the pulse-width modulation scheme for voltage regulation may be replaced by a linear voltage regulator if desired. These changes are well within the ambit of one skilled in the art and is deemed to be within the scope of this invention.

In operation, upon selection of the first swing height or speed setting, the control outputs the first voltage to the motor regardless of the swing height detected. However, in the event that the swing height exceeds the first predetermined swing height setting of, for example, greater than 9°, it is preferable for the control to cut-off the voltage applied to the motor for the duration of the portion of the swing that exceeds the first swing height setting.

Preferably, the first voltage is sufficient to enable the swing carriage to reach about 12°, a little beyond the first swing height setting to enable the prongs to interrupt light emitting from the IRLED.

If the second swing amplitude setting is selected, again the control outputs the first voltage to the motor until the swing height exceeds the first swing height setting of about 9°. Upon the swing height exceeding the first swing height setting, the control outputs the second voltage to the motor for the duration of the portion of the swing that exceeds the first swing height setting. In the second swing amplitude setting, the control outputs the second voltage which would enable the swing carriage to reach greater than 12°, for instance.

If the third swing height setting is selected, again the control outputs the first voltage to the motor until the swing height exceeds the first swing height setting of 9°. Upon the

swing height exceeding the first swing height setting, the control outputs the third voltage to the motor for the duration of the portion of the swing that exceeds the first swing height setting. The third voltage enables the swing carriage to reach greater than the second setting, but preferably less than 22° for instance.

In the second and third swing mode, however, when and if the swing height exceeds 22°, light emitting from the IRLED is again uninterrupted, causing the control to output the first voltage to the motor for the duration of the portion of the swing height that exceeds the third swing height setting to prevent excessively high swing amplitude. It should be noted that this can apply to the first mode. However, since the voltage supplied to the motor can be cut-off when the amplitude exceeds 9°, it will generally not occur, but adds additional protection, however.

Given the disclosure of the present invention, one versed in the art would readily appreciate the fact that there can be many other embodiments and modifications that are well within the scope and spirit of the disclosure set forth herein, but not specifically depicted and described. For example, although the present invention relates to a swing construction for an infant or child, the same teaching and principle may be applied to swings that handle a lighter object such as a doll, as well as for a heavier person such as an adult. Accordingly, all expedient modifications readily attainable by one versed in the art from the disclosure set forth herein that are within the scope and spirit of the present invention are to be included as further embodiments of the present invention. Accordingly, the scope of the present invention is to be as set forth in the appended claims.

What is claimed is:

- 1. A swing assembly comprising:
- a scat:
- at least one hanger connected to said seat;
- a support frame supporting said hanger, and
- a swing drive mechanism mounted on said support frame for oscillating said hanger relative to said support frame, said swing drive mechanism comprising:
 - an axle mounted on said support frame, wherein said 40 hanger is operatively connected to said axle;
 - a drive sleeve mounted coaxially and rotatably about said axle, wherein said sleeve is rotatable relative to said axle;
 - a drive flange mounted on said axle;
 - a drive flange coupling device positioned between said drive sleeve and said drive flange to cause said axle to oscillate with said drive sleeve;
 - a crank linked to said sleeve for oscillating said sleeve;
 - a motor for rotating said crank.
- 2. A swing assembly according to claim 1, wherein said coupling device comprises at least one spring mounted coaxially and rotatably relative to said axle and collinearly adjacent relative to said sleeve, wherein said spring is 55 positioned to enable engagement with said sleeve.
- 3. A swing assembly according to claim 2, wherein said coupling device further comprises a hub member rotatably mounted on said axle, wherein said spring is coaxially mounted to said hub member, said hub member including 60 abutments for engaging with said drive flange, whereby torque applied to said sleeve is transferred to said spring which causes said hub member to rotate and cause said abutments to engage said drive flange and transfer to said axle
- 4. A swing assembly according to claim 3, wherein said sleeve includes a channel running parallel with said axle and

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said crank has a ball mounted thereon, said ball being mounted in said channel and slideable and relative thereto, said ball being slideably movable and rotatable relative to said crank, whereupon rotation of said crank causes said sleeve to oscillate about said axle and along with said axle.

- 5. A swing assembly according to claim 4, wherein said motor has an output shaft mounted substantially perpendicular to said axle and said crank rotates about an axis that is perpendicular to said output shaft and said axle.
- 6. A swing assembly according to claim 1, further comprising a control for changing the swing amplitude.
- 7. A swing assembly according to claim 6, wherein said control has means for selectively providing at least two different predetermined swing amplitudes.
- 8. A swing assembly according to claim 7, wherein said control has means for selectively providing three different predetermined swing amplitudes.
- 9. A swing assembly according to claim 8, wherein said control has means for detecting the swing amplitude.
- 10. A swing assembly according to claim 9, wherein said control has means for controlling the swing amplitude based on the amplitude detected and the amplitude selected.
 - 11. An open top swing comprising:
 - a scat:

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- a pair of hangers connected to said seat;
- a free standing support frame pivotally supporting said hanger, said support frame comprising: a rear base:
 - first and second opposed legs extending upwardly at an angle from ends of said rear base;
 - a front base;
 - third and fourth opposed legs extending upwardly at an angle from ends of said front base,
 - wherein said first and third legs converge toward each other, and said second and fourth legs converge toward each other;
 - a first connector attached to said first and third legs for maintaining said first and third legs at a fixed position relative to each other;
 - a second connector attached to said second and fourth legs for maintaining said second and fourth legs at a fixed position relative to each other;
 - a first axle journaled for rotation on said first connector;
 - a second axle journaled for rotation on said second connector;
 - a hub mounted to each of said first and second axles, wherein one of said hangers is mounted to one of the hubs and the other of said hangers mounted to the other of said hubs; and
 - a swing drive mechanism mounted on said one of said first and second connector and operatively connected to respective one of said first and second axle for oscillating said seat, wherein said swing drive mechanism has means for selectively controlling the degree of rotation of said first and second axles.
- 12. An open top swing according to claim 11, wherein said front base is substantially trapezoidal shaped, defined by a median arm and a pair of oppositely extending arms extending from ends of said medial arm, wherein said third and fourth legs extend from ends of said oppositely extending arms
- 13. An open top swing according to claim 12, wherein said median arm is substantially parallel to said rear base and extends rearwardly toward said rear base.
- 14. An open top swing according to claim 11, further comprising an overrotation stop mounted to each of said hubs and a cooperating overrotation stop mounted to each of

said first and second connectors adjacent each of said hubs, wherein said stop and said cooperating stop prevent overrotation of said hubs relative to the first and second connectors.

- 15. An open top swing according to claim 11, wherein said 5 swing drive mechanism has control means for selectively providing at least two different predetermined swing amplitudes.
- 16. An open top swing according to claim 15, wherein said control means selectively provides three different predetermined swing amplitudes.
- 17. An open top swing according to claim 16, wherein said control means includes means for detecting the swing amplitude and controls the swing amplitude based on the amplitude detected and the amplitude selected.
- 18. An open top swing according to claim 17, wherein 15 said swing drive mechanism comprises:
 - a drive sleeve mounted coaxially and rotatably about said one axle connected to said one connector mounting said swing drive mechanism, wherein said sleeve is rotatable relative to said one axle;
 - a drive flange mounted on said one axle to provide a limited degree of rotation of said sleeve relative to said axle;
 - a crank linked to said sleeve for oscillating said sleeve; and 25
 - a motor fixedly connected relative to said one connector and operatively connected to said crank for rotating said crank, wherein said sleeve converts rotary motion to oscillatory motion to thereby oscillate said one axle and thus said one hub, thereby oscillating said seat via said hangers.
- 19. An open top swing according to claim 18, further comprising a drive flange coupling device positioned between said drive sleeve and said drive flange to cause said axle to oscillate with said drive sleeve.
- 20. An open top swing according to claim 19, wherein said coupling device comprises at least one spring mounted coaxially and rotatably relative to said axle and collinearly adjacent relative to said sleeve, wherein said spring is positioned to enable engagement with said sleeve.
- 21. An open top swing according to claim 20, wherein said coupling device further comprises a hub member rotatably mounted on said axle, wherein said spring is coaxially mounted to said hub member, said hub member including abutments for engaging with said drive flange, whereby torque applied to said sleeve is transferred to said spring which causes said hub member to rotate and cause said abutments to engage said drive flange and transfer to said axle
- 22. An open top swing according to claim 21, wherein said sleeve includes a channel running parallel with said axle and said crank has a ball mounted thereon, said ball being mounted in said channel and slideable and relative thereto, said ball being slideably movable and rotatable relative to said crank, whereupon rotation of said crank causes said sleeve to oscillate about said axle and along with said axle.
- 23. A swing drive mechanism adapted for a swing that includes a supporting frame and an axle operatively connected to a hanger suspending a seat comprising:
 - a drive sleeve adapted for mounting coaxially and rotatably about said axle, wherein said sleeve is rotatable relative to said axle;
 - a drive flange adapted for mounting on said axle;
 - a drive flange coupling device positioned between said 65 drive sleeve and said drive flange and adapted to cause said axle to oscillate with said drive sleeve;

a crank linked to said sleeve for oscillating said sleeve;

a motor for rotating said crank.

- 24. A swing drive mechanism according to claim 23, wherein said coupling device comprises at least one spring adapted for mounting coaxially and rotatably relative to said axle and collinearly adjacent relative to said sleeve, wherein said spring is positioned to enable engagement with said sleeve.
- 25. A swing drive mechanism according to claim 24, wherein said coupling device further comprises a hub member adapted for rotatably mounting on said axle, wherein said spring is coaxially mounted to said hub member, said hub member including abutments for engaging with said drive flange, wherein torque applied to said sleeve is transferred to said spring which causes said hub member to rotate and cause said abutments to engage said drive flange and transfer to said axle.
- 26. A swing drive mechanism according to claim 25, wherein said coupling device comprises two springs coaxially mounted to said hub member, wherein said springs are arranged so that said sleeve can engage one of the two springs and said drive flange can engage the other of said two springs when said sleeve is rotated in one direction, and said sleeve can engage said other spring and said drive flange can engage said one spring when said sleeve is rotated in the opposite direction.
- 27. A swing drive mechanism according to claim 26, wherein said springs are coiled in opposite directions such that said sleeve and drive flange tend to cause said springs to coil tighter around said hub member, wherein said springs, said hub member and said drive flange provide three spring gradients.
- 28. A swing drive mechanism according to claim 27, wherein said sleeve is freely rotatable relative to said springs for a limited degree, wherein the free limited degree rotation provides first of said three spring gradients, wherein said sleeve engages one of said springs and the other of said springs engages said drive flange upon rotation of said sleeve beyond said free rotation, causing said two springs to be active, providing second of said three spring gradients, wherein further rotation of said sleeve rotates said hub member along with said sleeve and causes said abutments to engage said drive flange which prevents said hub member from rotating relative to said drive flange, causing said spring engaging said drive flange to be inactive, providing the third spring gradient.
- 29. A swing drive mechanism according to claim 28, wherein said sleeve includes a channel adapted to run parallel with said axle and said crank has a ball mounted thereon, said ball being mounted in said channel and slideable and relative thereto, said ball being slideably movable and rotatable relative to said crank, whereupon rotation of said crank causes said sleeve to oscillate about said axle and along with said axle.
- 30. A swing drive mechanism according to claim 29, wherein said crank has an offset driven portion which extends a distance from its axis of rotation, wherein said ball is mounted on said offset portion and orbits about said axis of rotation.
- 31. A swing drive mechanism according to claim 30, wherein said motor has an output shaft mounted substantially perpendicular to said axle and said crank rotates about said axis that is perpendicular to said output shaft.
- 32. A swing drive mechanism according to claim 31, further comprising control means adapted for selectively controlling the degree of rotation of said axle.

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- 33. A swing drive mechanism according to claim 32, wherein said control means has means for selectively providing three predetermined different swing amplitudes and includes means for detecting the swing amplitude, wherein said control means controls the swing amplitude based on 5 the amplitude detected and the amplitude selected.
- 34. An open top support frame for a swing having a pair of hangers suspending a seat comprising:
 - a rear base:
 - first and second opposed legs extending upwardly at an angle from ends of said rear base;
 - a front base:
 - third and fourth opposed legs extending upwardly at an angle from ends of said front base, wherein said first 15 and third legs converge toward each other, and said second and fourth legs converge toward each other;
 - a first connector attached to said first and third legs for maintaining said first and third legs at a fixed position relative to each other;
 - a second connector attached to said second and fourth legs for maintaining said second and fourth legs at a fixed position relative to each other;
 - a pivot operatively mounted to said first connector;
 - a second pivot operatively mounted to said second connector:
 - a hub mounted to each of said first and second pivots, wherein one of said hangers is mounted to one of the hubs and the other of said hangers mounted to the other of said hubs; and
 - an overrotation stop mounted to each of said hubs and a cooperating overrotation stop mounted to each of said first and second connectors adjacent each of said hubs, wherein said stop and said cooperating stop prevent 35 overrotation of said hubs relative to the first and second connectors.
- 35. An open top support frame according to claim 34, wherein said front base is substantially trapezoidal shaped, defined by a median arm and a pair of oppositely extending 40 arms extending from ends of said medial arm, wherein said third and fourth legs extend from ends of said oppositely extending arms.
- 36. An open top support frame according to claim 35, wherein said median arm is substantially parallel to said rear 45 base and extends rearwardly toward said rear base.
- 37. A method of selectively controlling swing heights or amplitudes in a swing that has a motor operated swing drive mechanism comprising the steps of:
 - providing a selection of at least first and second swing 50 height settings, wherein said first setting is smaller than said second setting;
 - selectively inputting at least one of no voltage, a predetermined first voltage and a predetermined second voltage to said motor based on the selection of the swing height setting to achieve the selected swing height, wherein said first and second voltages are

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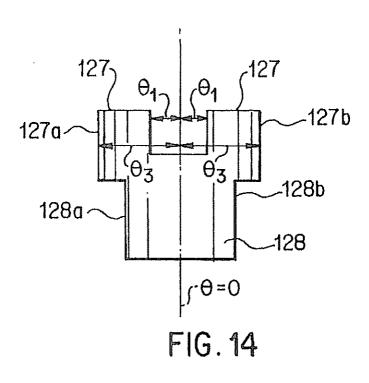
higher than zero, and said first voltage is lower than said second voltage.

- 38. A method according to claim 36, wherein upon selection of said first swing height setting, applying said first voltage to said motor.
- 39. A method according to claim 37, further comprising the step of detecting the swing amplitude.
- 40. A method according to claim 39, wherein upon selection of said first swing height setting, comprising the steps of:
 - initially applying said first voltage to said motor;
 - continuously maintaining said first voltage to said motor until the swing height is greater than said selected first swing height;
 - applying no voltage to said motor when and if the sensed swing height exceeds said first height setting for the duration of the portion of the swing that exceeds said first swing height.
- 41. A method according to claim 39, wherein upon selection of said second swing height setting, comprising the steps of:
 - initially applying said first voltage to said motor;
 - continuously maintaining said first voltage to said motor until the swing height is greater than said first swing height setting;
 - applying said second voltage to said motor for the duration of the portion of the swing that is greater than said first swing height setting.
 - 42. A method according to claim 39, further comprising the steps of:
 - further providing a third swing height setting, wherein said third setting is greater than said second setting; and
 - selectively inputting at least one of said no voltage, said predetermined first voltage, said predetermined second voltage and a third predetermined voltage to said motor based on the selection of the swing height setting to achieve the selected swing height, wherein said third voltage is higher than zero voltage and higher than said second voltage.
 - 43. A method according to claim 42 further comprising the steps of, upon selection of said third swing height setting:
 - initially applying said first voltage to said motor;
 - continuously maintaining said first voltage to said motor until the swing height is greater than said first swing height setting;
 - applying said third voltage to said motor for the duration of the portion of the swing that is greater than said first swing height setting.
 - 44. A method according to claim 43, further comprising the step of, when and if the swing height is greater than said selected third swing height setting, applying said first voltage to said motor for the duration of the portion of the swing that is greater than said third swing height setting.

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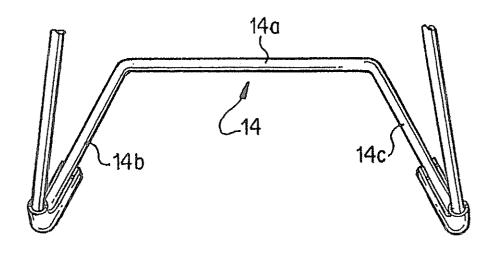


FIG.1A

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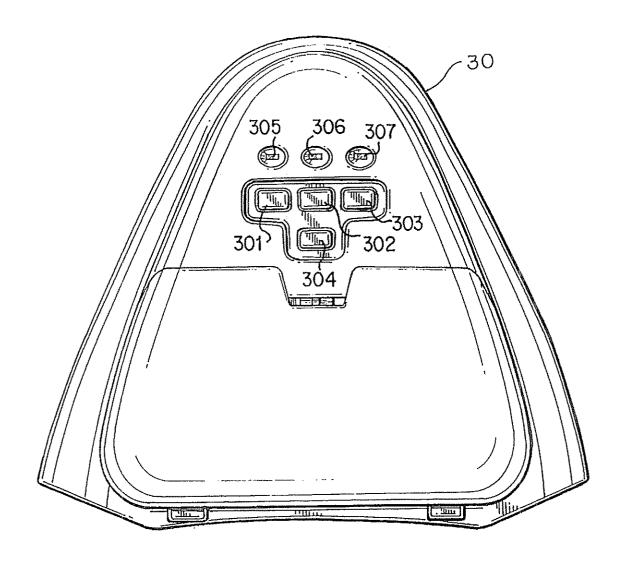
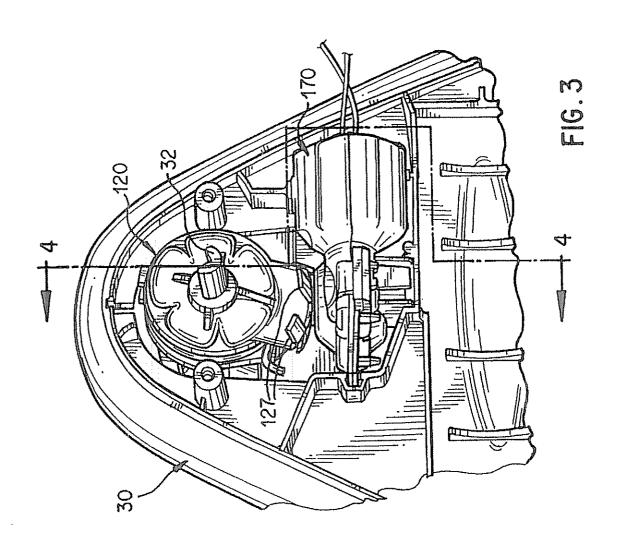


FIG. 2

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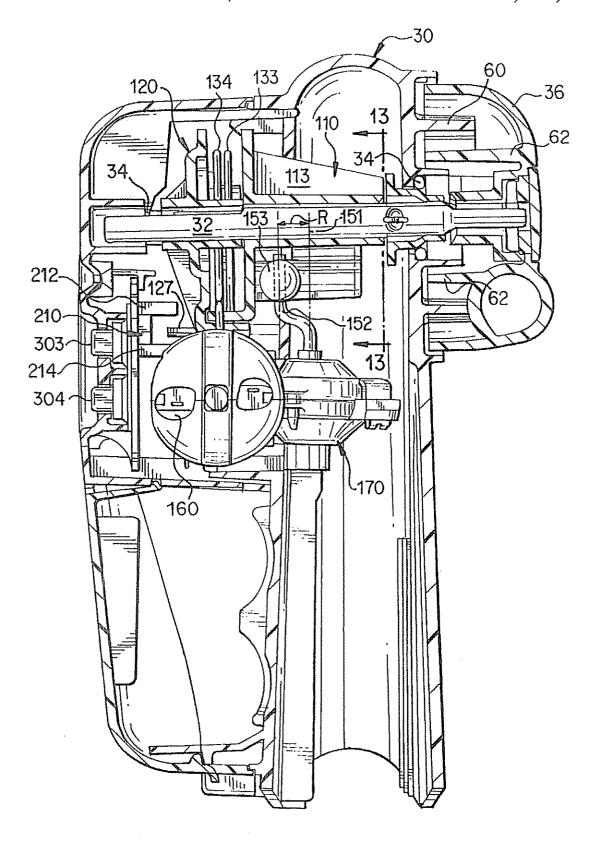
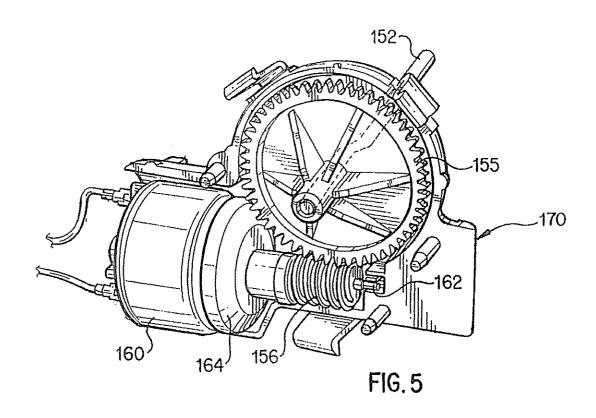


FIG.4

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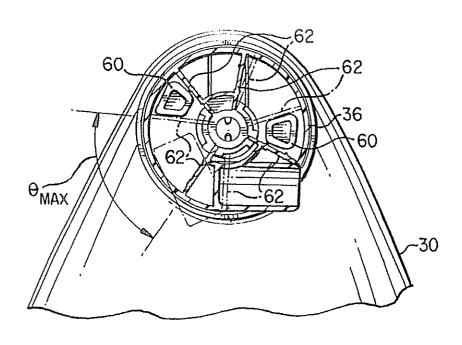
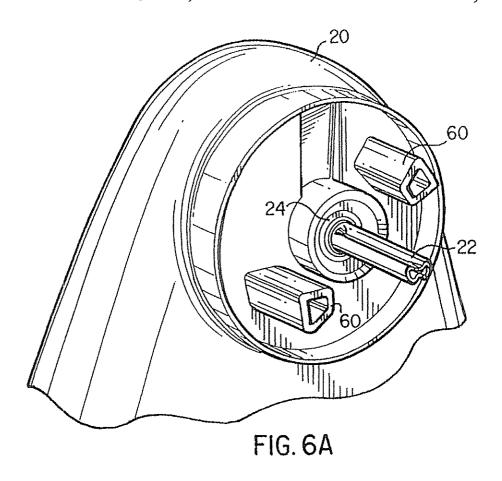
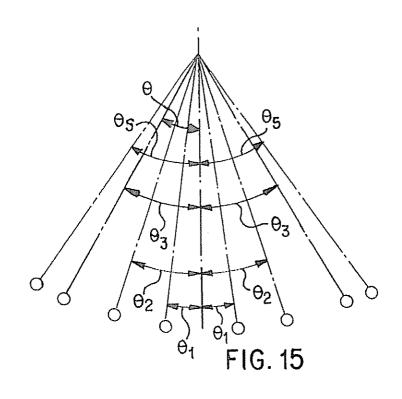


FIG. 6

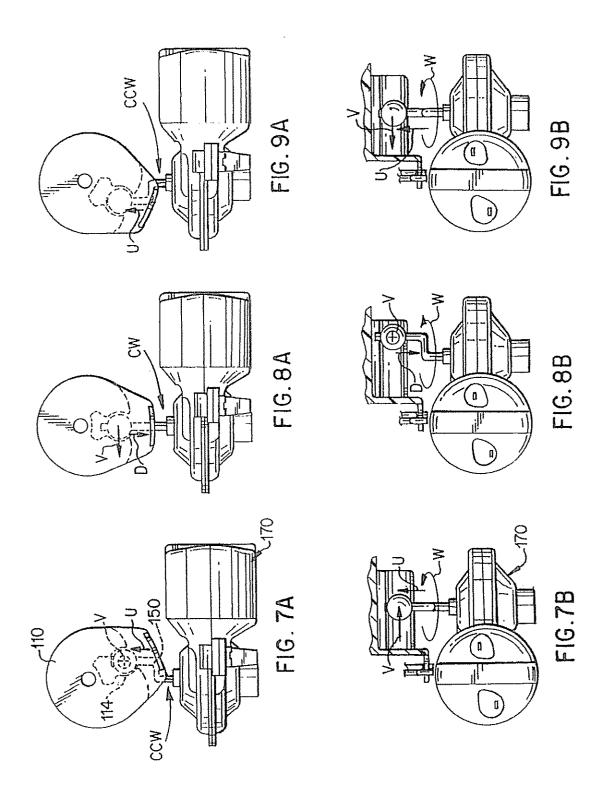
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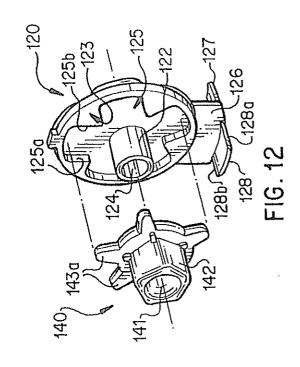
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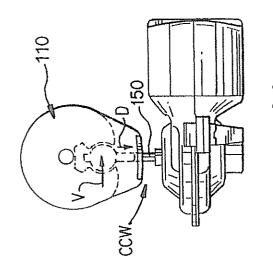
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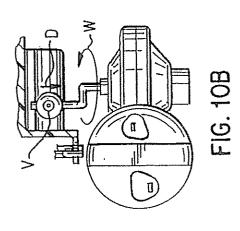


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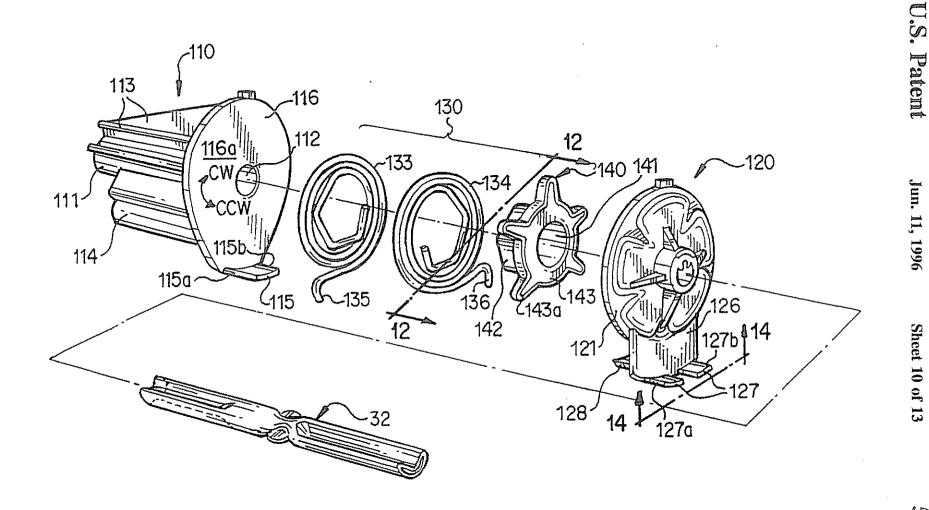


FIG. 11

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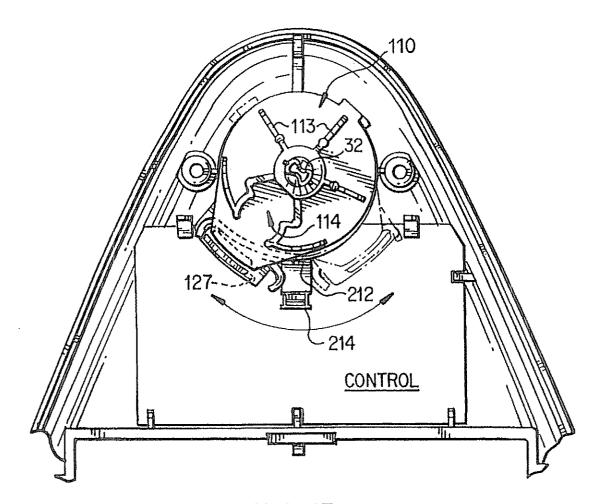
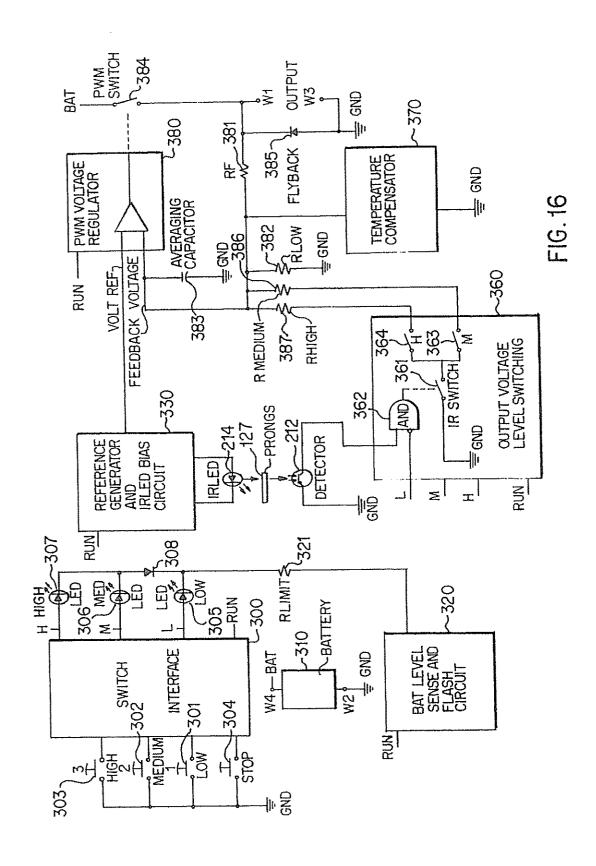


FIG. 13



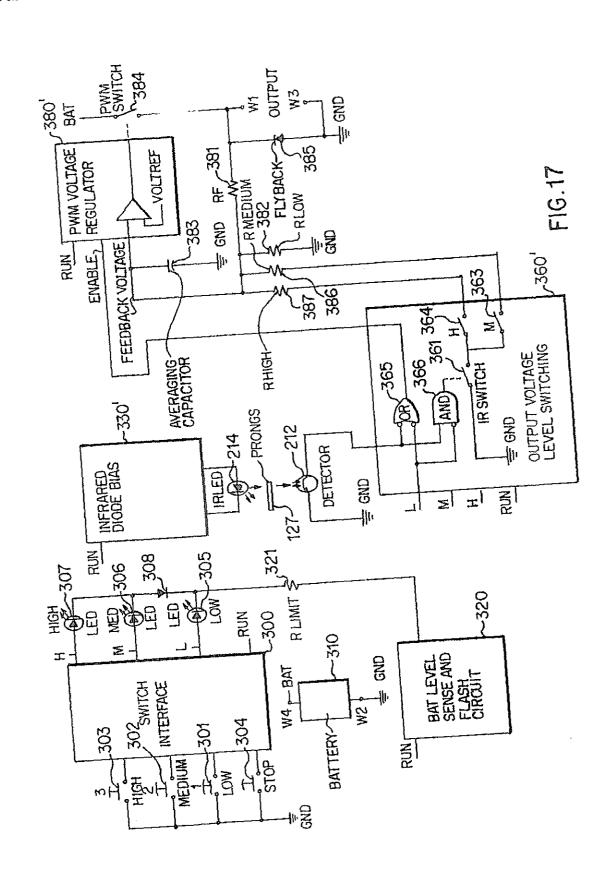


EXHIBIT "B"

(12) United States Patent

Longenecker et al.

(10) Patent No.:

US 6,896,624 B2

(45) Date of Patent:

May 24, 2005

(54) FOLDABLE SWING HAVING ROTATABLE HANDLE

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Exton, PA (US)

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(51) Int. (CI.7		A63G	9/16
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- Field of Search 472/118–125; (58)297/273, 274, 281, 277, 354.12, 365, 376

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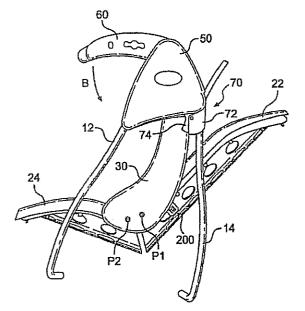
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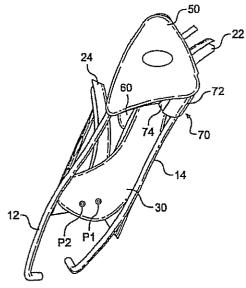
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ABSTRACT

A child swing includes a swing frame and a swing handle rotationally coupled to the swing frame such that the swing handle may rotate about a handle rotational axis between at least two positions. The swing may further include a swing seat and at least one hanger arm rotationally coupled to the swing frame and supporting the swing seat. The at least two positions may include a lift position wherein a central portion of the swing handle is arranged above the rotational axis, an open access position wherein a central portion of the swing handle is arranged rearward of the rotational axis, an entertain position wherein a central portion of the swing handle is arranged forward of the rotational axis, and a storage position wherein a central portion of the swing handle is arranged forward and below the rotational axis.

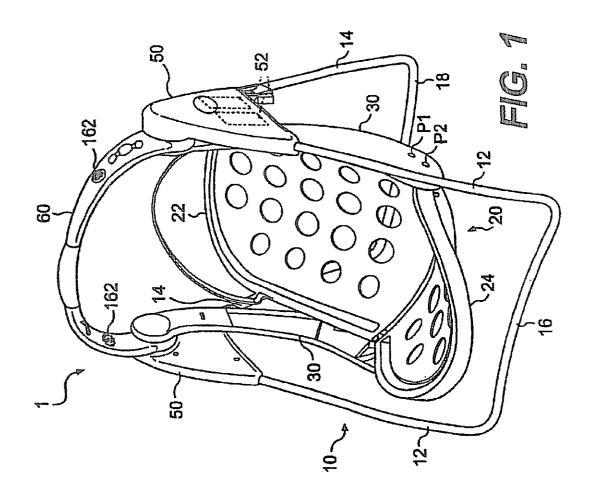
46 Claims, 17 Drawing Sheets





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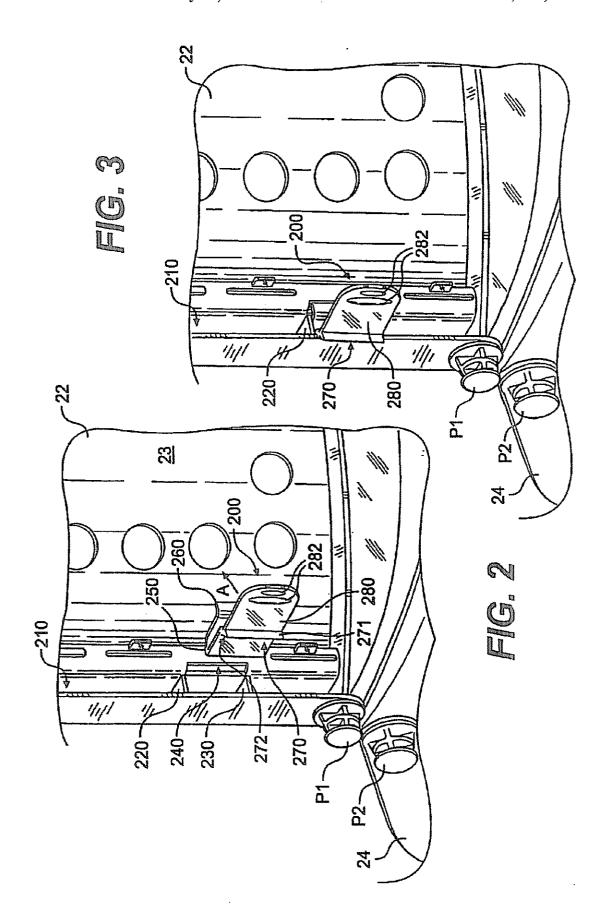


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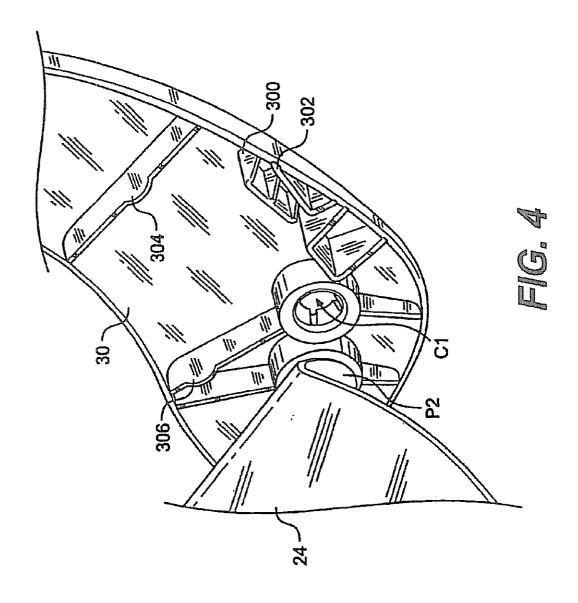
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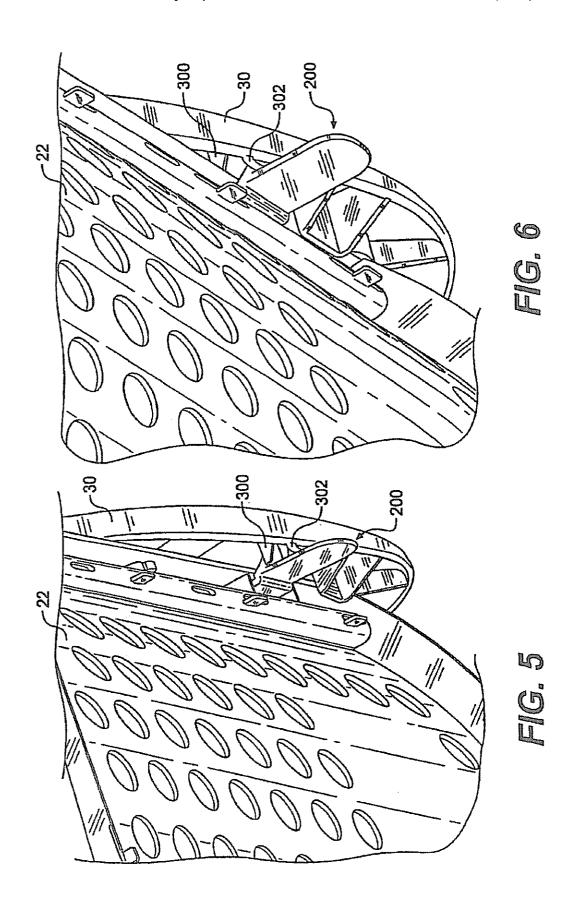
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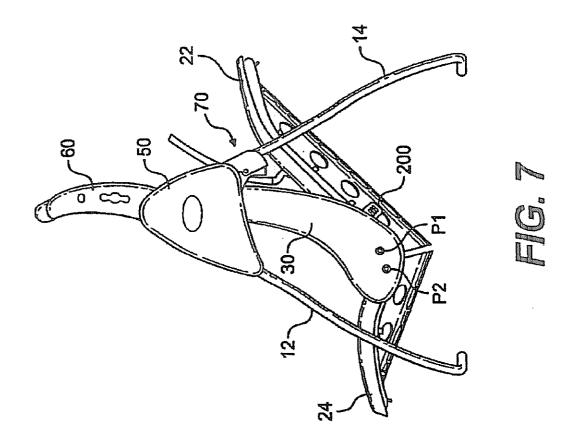
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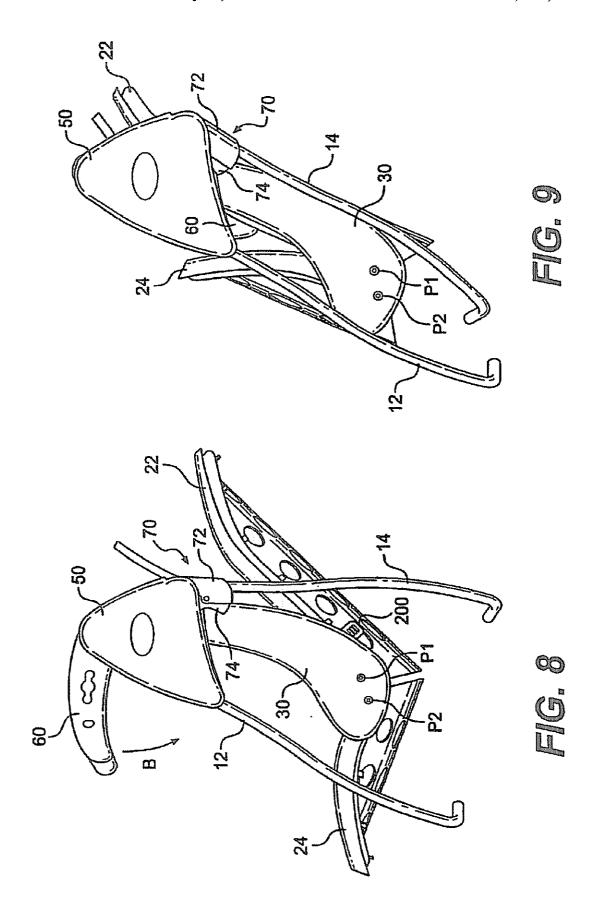
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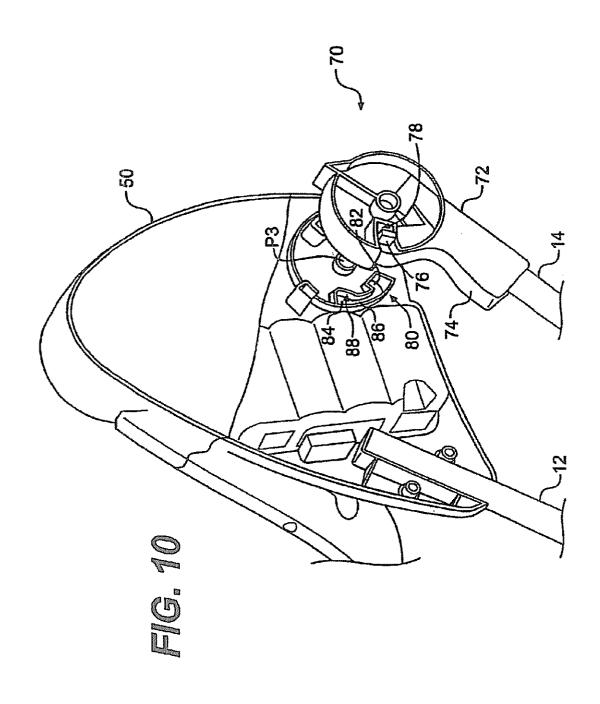
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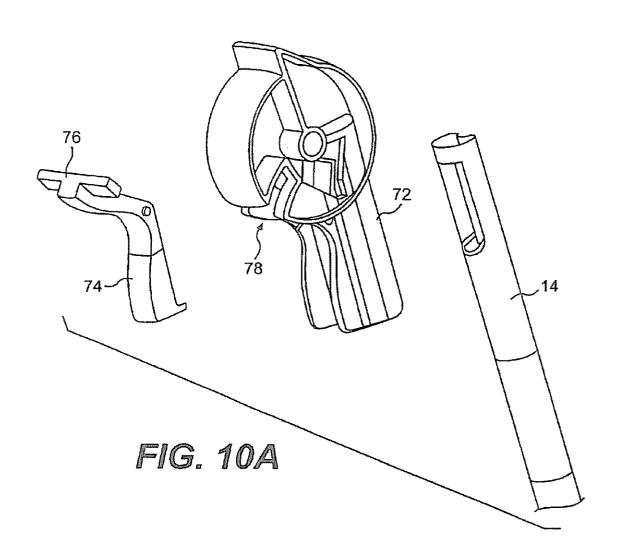
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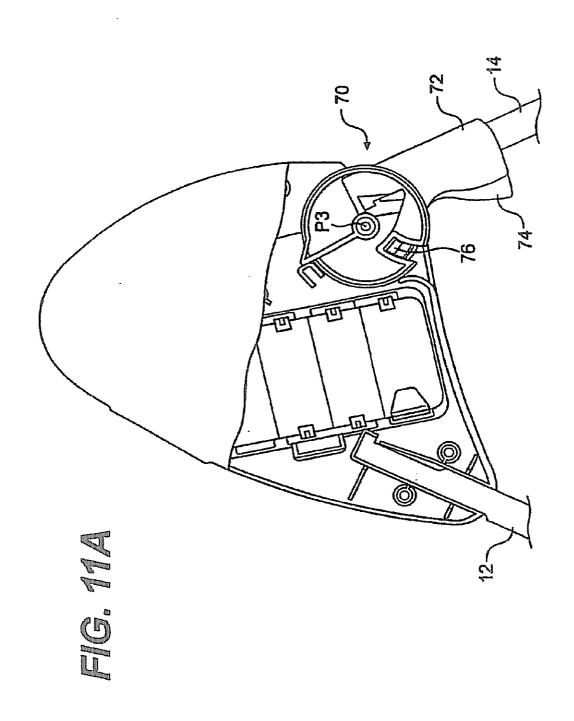
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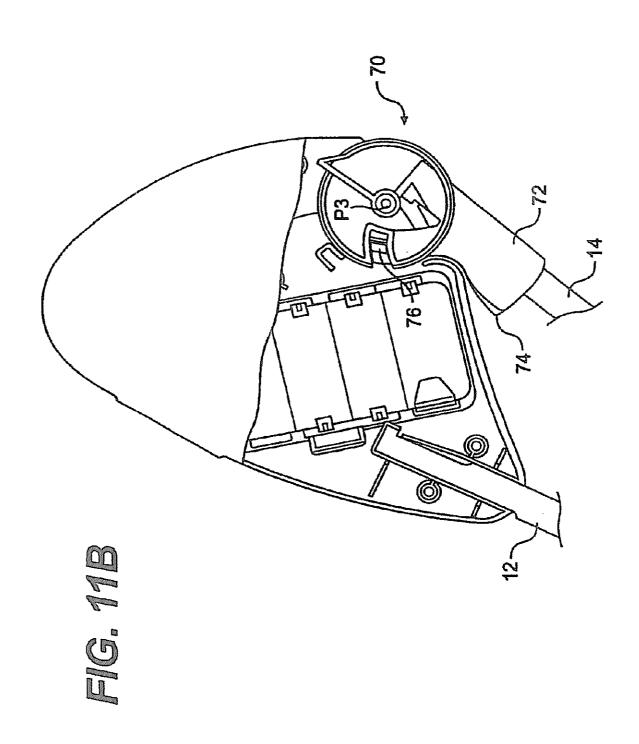
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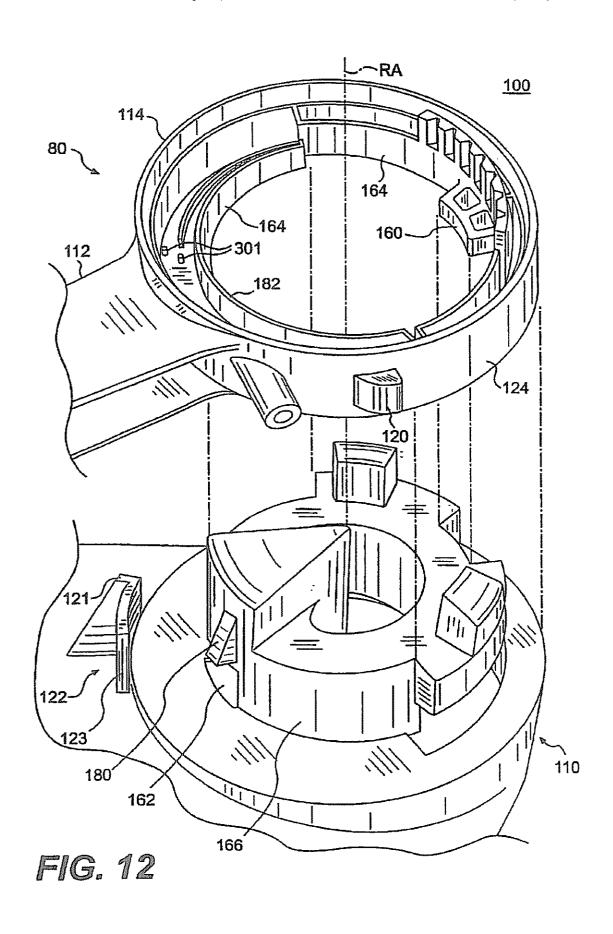
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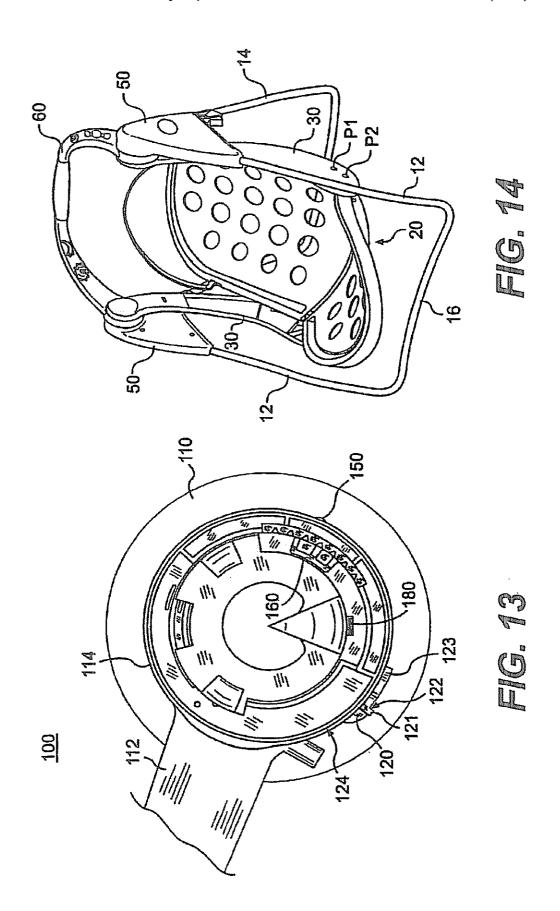
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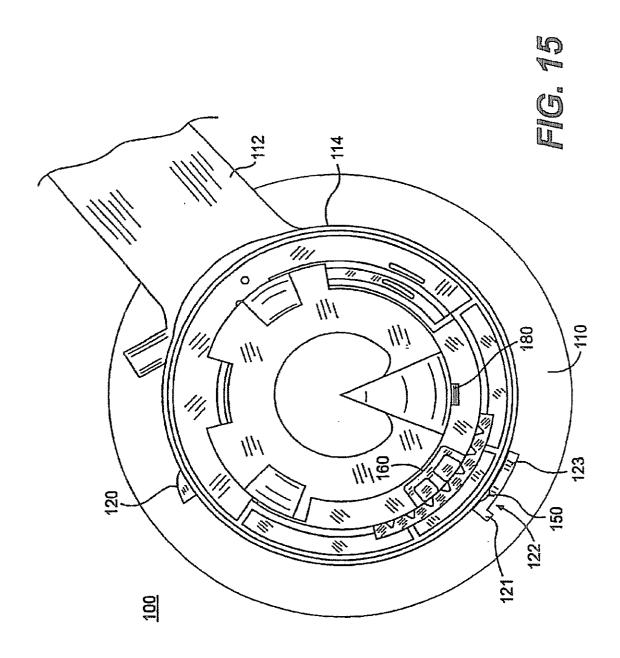
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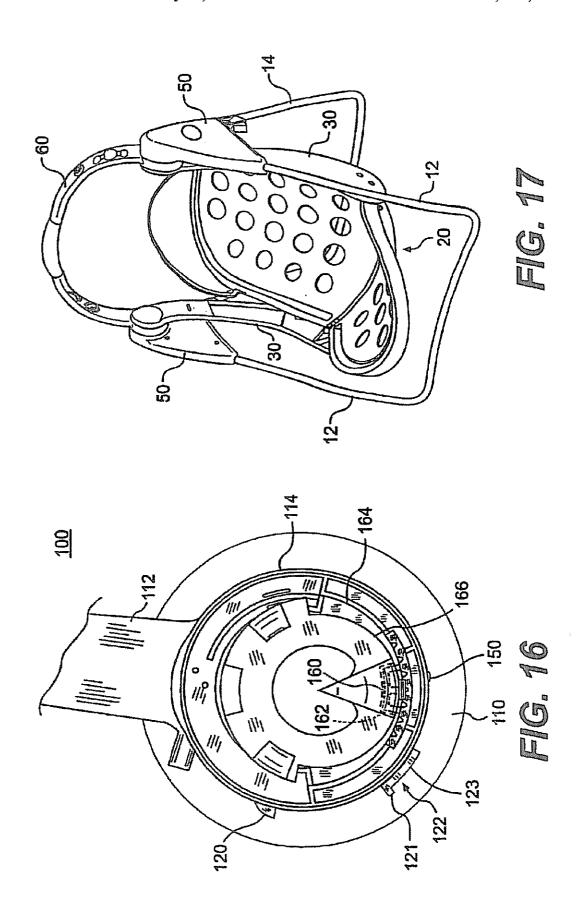
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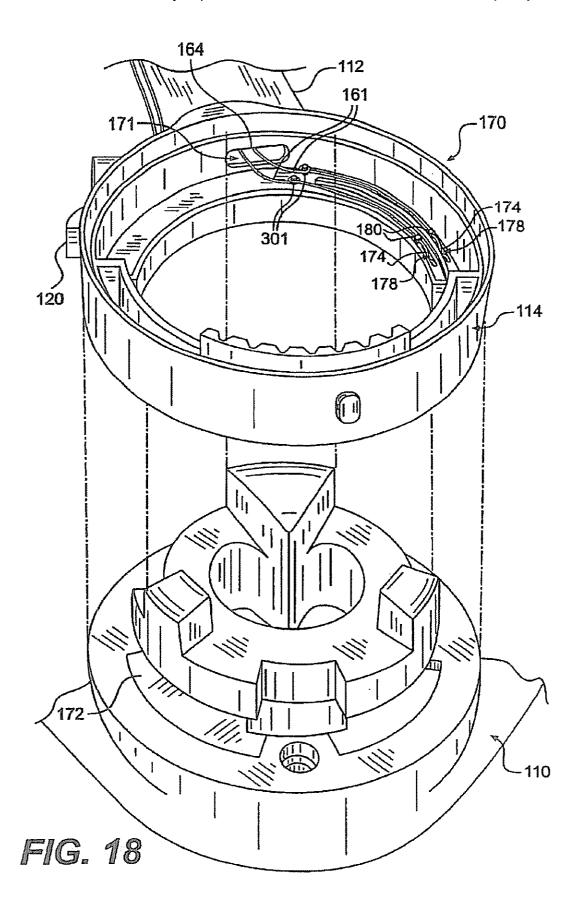
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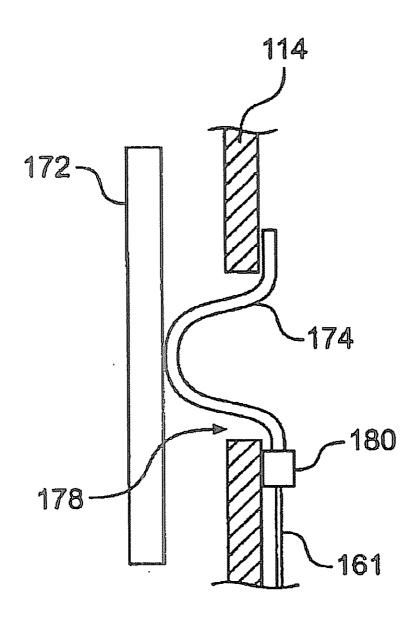
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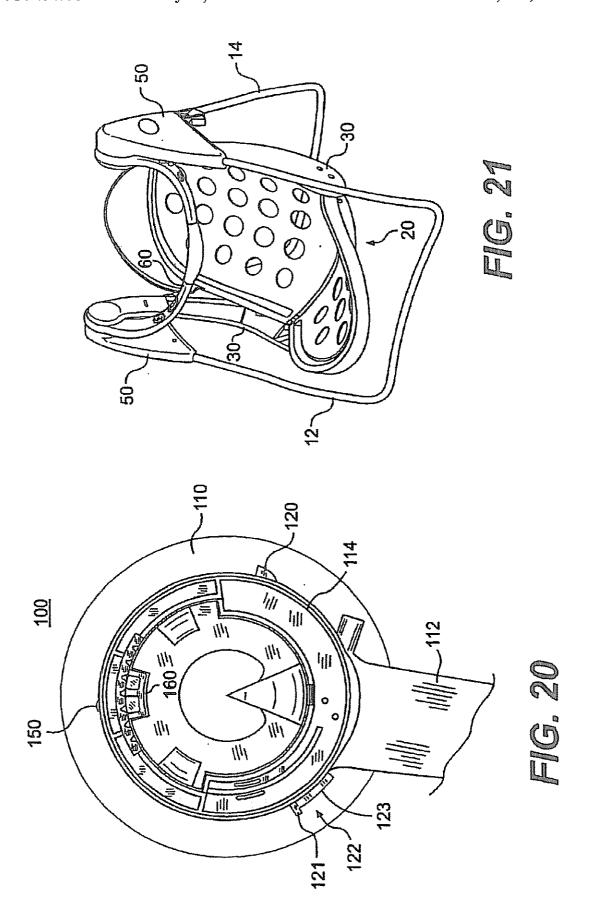
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FOLDABLE SWING HAVING ROTATABLE HANDLE

FIELD OF THE INVENTION

This invention relates to a rotatable handle for a swing. More specifically, this invention relates to a rotatable handle for a foldable child swing.

BACKGROUND OF THE INVENTION

Various types of child or infant swings are known in the art. Typically, such swings include a support frame, hanger arms pivotably attached to the support frame, and a seat attached to the hanger arms.

It is also known to include a handle on the swing to allow a user to carry the swing. The handle is rigidly attached to the support frame, but the swing seat can sway back-andforth as the swing is carried, which makes the swing somewhat awkward to move.

SUMMARY OF THE INVENTION

An aspect of the present invention relates to a swing that comprises a swing frame and a swing handle rotationally coupled to the swing frame such that the swing handle may rotate about a handle rotational axis between at least two positions. The swing further may comprise a swing seat and at least one hanger arm rotationally coupled to the swing frame and supporting the swing seat. The at least two positions may include a lift position wherein a central portion of the swing handle is arranged above the rotational axis, an open access position wherein a central portion of the swing handle is arranged rearward of the rotational axis, an entertain position wherein a central portion of the swing handle is arranged forward of the rotational axis, and a storage position wherein a central portion of the swing handle is arranged forward and below the rotational axis.

According to the present invention, in the lift position, the swing handle may be locked to prevent rotation of the swing handle relative to the swing frame. In the open access 40 position, the swing handle may be stopped from any further rearward rotation relative to the swing frame. In the entertain position, a detent mechanism may frictionally resist rotational motion by the swing handle in a forward or rearward direction relative to the swing frame. In the storage position, 45 the swing handle may be nested between a seat back and a seat bottom of the swing when the swing is folded.

The swing also may comprise a handle support structure supported by the swing frame, and the swing handle may comprise a handle portion and a support interface portion. 50 The support interface portion may be attached to the handle support structure. In addition, the handle portion may comprise at least one electronic device and electrical wiring extending from the at least one electronic device to the support interface portion. The electronic device may be a 55 light producing device and/or a sound producing device. The support interface portion and the handle support structure may include at least one moving contact assembly to provide electrical contact between the handle support structure and the support interface portion, and the electrical wiring may 60 electrically contact the at least one moving contact assembly. The moving contact assembly may be configured to maintain electrical contact over a rotational angle range as the swing handle rotates relative to the handle support structure, and this rotational angle range may correspond to the 65 position. angular range about the entertain position of the swing handle.

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It is to be understood that both the foregoing general description and the following detailed description are exemplary and explanatory only and are not restrictive of the invention, as claimed.

BRIEF DESCRIPTION OF THE DRAWINGS

The accompanying drawings, which are incorporated in and constitute a part of this specification, illustrate several embodiments of the invention and, together with the description, serve to explain the principles of the invention.

- FIG. 1 is a front perspective view of a swing, where the swing handle is in an entertain position, in accordance with the invention.
- FIG. 2 is a rear perspective, exploded, detail view of the seat back and latch of the swing seat.
 - FIG. 3 is a rear perspective, detail view of the seat back and latch of the swing seat.
- FIG. 4 is a rear perspective, detail view of the inner surface of a hanger arm of the swing.
 - FIG. 5 is a rear perspective, detail view of a hanger arm and seat back, where the seat back is in a first in-use position.
- FIG. 6 is a rear perspective, detail view of the hanger arm and seat back, where the seat back is in a second in-use position.
 - FIG. 7 is a side view of the swing in the first in-use position.
 - FIG. $\bf 8$ is a side view of the swing in an intermediate fold position.
 - FIG. 9 is a side view of the swing in a fully folded position.
 - FIG. 10 is an exploded, detail view of the rear leg fold mechanism.
 - FIG. 10A is an exploded, perspective view of the leg socket member and the release lever of the rear leg fold mechanism.
 - FIG. 11A is a detail view of the rear leg fold mechanism in a locked, in-use position.
 - FIG. 11B is a detail view of the rear leg fold mechanism in a fold position.
 - FIG. 12 is an exploded, detail view of a swing handle assembly in accordance with the invention.
 - FIG. 13 is a detail view of the right-side, swing handle assembly, where the swing handle is rotated to an open access position.
 - FIG. 14 is a front perspective view of the swing, where the swing handle is in the open access position.
 - FIG. 15 is a detail view of the right-side, swing handle assembly, where the swing handle is rotated to an entertain position.
 - FIG. 16 is a detail view of the right-side, swing handle assembly, where the swing handle is rotated to a lift position.
 - FIG. 17 is a front perspective view of the swing, where the swing handle is in the lift position.
 - FIG. 18 is an exploded, detail view of a swing handle assembly including a moving contact assembly in accordance with the invention.
 - FIG. 19 is a side view of a spring contact and an arc shaped contact according to an embodiment of the invention.
 - FIG. 20 is a detail view of the right-side, swing handle assembly, where the swing handle is rotated to a storage position.
 - FIG. 21 is a front perspective view of the swing, where the swing handle is rotated to the storage position.

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DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

Reference will now be made in detail to presently preferred embodiments of the invention, examples of which are illustrated in the accompanying drawings. An effort has been made to use the same reference numbers throughout the drawings to refer to the same or like parts.

FIG. 1 illustrates a swing 1 according to an exemplary embodiment of the present invention. The swing 1 has a foldable frame and a reclinable swing seat 20 that can be moved between a fold position, a first, upright in-use position, and a second, reclined in-use position. The swing 1 also includes a handle 60 that can be used to carry the swing. The handle 60 can be rotated between at least two positions. For example, in one embodiment, the handle 60 can be rotated between four positions, including: an open access position during which a child can be seated in the swing, a lift and lock position during which the swing can be carried by the handle, an entertain position during which a child seated in the swing can view lights and/or other play features of the handle, and a storage position during which the handle can be compactly stored when the frame is folded. The number of positions may also be more than four.

The swing 1 generally includes a support frame 10, a seat 20 having a seat back 22 and a seat bottom 24, and pair of hanger arms 30 that connect the seat 20 to the support frame 10. The seat back 22 is pivotally connected to the hanger arms 30 at pivots P1, and the seat bottom 24 is pivotally connected to the hanger arms 30 at pivots P2.

The support frame 10 generally includes front legs 12, rear legs 14, a front cross member 16 extending between the front legs 12, a rear cross member 18 extending between the rear legs 14, and first and second housings 50. In the illustrated embodiment, the front legs 12 of the support frame 10 are fixedly connected to the respective housings 50, and the rear legs 14 of the support frame 10 are pivotally connected to the respective housings 50 to allow the swing 1 to fold, as will be explained below. Alternatively, the front legs can be pivotally connected to the respective housings, $_{40}$ and the rear legs can be fixedly connected to the respective housings. In a further embodiment, both the front and rear legs can be pivotally connected to the housings. The fold swing operation will be described below in connection with FIGS. 7-9.

The support frame 10, in addition to supporting the support hangers 30 and ultimately the seat 20, also supports a swing handle 60, which is part of a swing handle assembly. The swing handle 60 is rotationally connected at either end to the housings 50 and to the hanger arms 30. The operation 50 and second ribs 300, 302 corresponding to the first, upright and structure of the swing handle assembly will be described below in connection with FIGS. 12-21.

The swing 1 also can include a power supply 52 within one of the housings 50 for supplying power to a motor to drive the motion of the swing 1 and/or for supplying power 55 to the swing's electronic devices. The power supply 52 may comprise, for example, a battery holder for holding batteries.

As mentioned above, the swing seat 20 can be moved between a fold position, a first, upright in-use position, and back 22 of the swing seat 20 is positionable in a first, upright in-use position, in a second, reclined in-use position in which the seat back 22 is adjusted rearward relative to its first in-use position, and in a fold position in which the seat back 22 is adjusted forward relative to its first, upright in-use 65 position. While the frame 10 of the swing 1 is in-use, that is, erect, the seat back 22 can be positioned in its first and

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second in-use positions, and, when the frame 10 is folded for storage, the seat back 22 can be positioned in its fold position.

The seat recline mechanism will now be described in connection with FIGS. 2-6. The scat recline mechanism includes a latch 200 positioned on each side of the seat back 22 for engagement with the respective hanger arms 30. FIG. 2 is a detail view of the right side of the seat back 22 and the latch 200. Although the figures generally show the structural relationship between the seat 20 and the hanger arms 30 by reference to only one hanger arm 30, it will be understood that, in the illustrated embodiment, the seat-hanger arm relationship on the left and right sides of the swing are mirror images.

The seat back 22 includes a channel 210 molded along at least a portion of the perimeter of the rear surface 23 of the seat back 22. Upper and lower ribs 220, 230 are positioned in the channel 210 for capturing the latch 200 therebetween. The seat back 22 also includes a slot 240 adjacent the ribs 220, 230 for receipt of a portion of the latch 200. As seen in FIG. 2, the latch 200 is spaced from the pivot P1.

The latch 200 has a U-shaped segment 250 configured for insertion into the channel 210 between ribs 220, 230, a flange 260 at one end of the U-shaped segment 250, and a locking ridge 270 at the other end of the U-shaped segment 250. The flange 260 is configured for insertion into slot 240 in a snap fit to secure the latch to the seat back 22. FIG. 3 illustrates the latch 200 positioned in the channel 210 of the seat back 22, with flange 260 extending through slot 240. The locking ridge 270 is configured to engage latchreceiving members, such as ribs, on the hanger arms 30 to secure the seat back 22 in a selected in-use position.

The latch 200 also includes a segment 280 with finger bumps 282. A user can press on the finger bumps 282 to flex the latch 200 inwardly, about the U-shaped segment 250, in the direction of arrow A in FIG. 2 to disengage the latch from the hanger arms 30. This segment 280 may be visible to the user to facilitate positioning of the seat back to a desired in-use position.

FIG. 4 shows the inner surface of the left-side hanger arm 30 in accordance with the invention, the right-side hanger arm 30 being a mirror image. The hanger arm 30 includes two through holes, one of which is labeled C1, to receive the pivot P1 of the seat back 22 and the pivot P2 of the seat bottom 24, respectively. In other embodiments, the seat back 22 and the seat bottom 24 can share a common pivot, and the hanger arm can include a single hub or single through hole to receive the pivot. The hanger arm 30 also includes first in-use position and the second, reclined in-use position, respectively. To position the seat back 22 in the first, upright in-use position, the locking ridge 270 of each latch 200 is positioned against the first rib 300 of the respective hanger arm 30, as shown in FIG. 5. To position the scat back 22 in the second, reclined in-use position, the locking ridge 270 of each latch 200 is positioned against the second rib 302 of the respective hanger arm 30, as shown in FIG. 6.

As can be seen from FIG. 2, the locking ridge 270 has a a second, reclined in-use position. More specifically, the seat 60 flat surface 271 and an angled surface 272. Due to the configuration of the locking ridge 270, the user only needs to actuate the latch 200 to move the seat back 22 in a rearward direction, for example, from the upright in-use position to the reclined in-use position. The user need not actuate the latch 200 to move the seat back 22 in a forward direction, for example, from the reclined in-use position to the upright in-use position. When moving the seat back 22

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forward from the reclined in-use position toward the upright in-use position, the user can grip the seat back 22 and pivot it forward, which causes the angled surface 272 to ride over the first rib 300. The latch 200 flexes inward until it passes the rib 300 and then restores to its at-rest configuration. At 5 this point, the user can release the seat back 22 to allow the flat surface 271 to rest on the first rib 300. The seat back 22 can be moved from the upright in-use position to the fold position in the same manner, that is, by simply gripping the seat back 22 and pivoting it forward.

In addition to ribs 300, 302, each hanger arm 30 can include a rib 304 on its inner surface, forward of rib 302, to maintain the seat back 22 in the fold position. The rib 304 has a detent so that, when folded, the side of the seat back 22 comes into contact with the detent in a friction fit and is maintained in the fold position until the user pushes the seat back 22 rearward, away from the rib 304. Similarly, the hanger arms 30 each can include a rib 306 with a detent for engagement with a side of the seat bottom 24 to maintain the seat bottom 24 in a fold position when pivoted upwardly and 20 rearwardly, toward the seat back 22.

Although only two in-use positions are shown in the figures, it will be understood that the hanger arms 30 can include additional ribs representative of additional in-use positions. Further, although the figures show the seat 20 suspended from a pair of hanger arms 30, the present invention envisions a swing having a single hanger arm to suspend the seat.

The latches 200 can be integrally molded as a unitary body with the seat back 22, or they can be fabricated separately from the seat back 22 and later releasably attached to the seat back 22 as shown in FIG. 2. When fabricated separately, the latches 200 can have a different color than the seat back 22 to make them more visible to the user. In addition, the latches 200 can be fabricated from any suitable material, including plastic (such as acetal), steel, and aluminum. When the latch 200 is fabricated from a relatively rigid material, such as a metal or metal alloy, as opposed to a flexible plastic, the latching force achieved by deformation and restoration of the plastic latch material can be accomplished by a spring appropriately located relative to the metal latch 200 and the channel 210.

The swing fold operation will now be described in connection with FIGS. 7–9. FIG. 7 shows the swing in an in-use position, the seat back 22 in its first, reclined in-use position, and the swing handle 60 in its lift and lock position. A rear leg fold mechanism 70 is mounted to each rear leg 14 adjacent each housing 50. To fold the swing 1, the rear leg fold mechanisms 70 are actuated by the user so that the rear legs 14 can pivot relative to the housings 50 toward the front legs 12.

FIGS. 10, 10A, 11A, and 11B illustrate the rear leg fold mechanism in more detail. The rear leg fold mechanism generally includes a leg socket member 72 to which the rear leg is mounted, a release lever 74, and a locking pin 76 connected to the release lever 74. As shown in FIG. 10A, the locking pin 76 of the illustrated embodiment is formed as part of the release lever 74; however, it will be understood that the locking pin 76 can be molded separately from, and 60 then connected by a suitable fastener to, the release lever 74. The socket member 72 is rotationally mounted to the respective housing 50 about pivot P3 and rotates with the rear leg 14 between the in-use position and the fold position. The locking pin 76 of the release lever 74 is configured to be 65 captured in slots 78, 80 in the socket member 72 and the housing 50, respectively. The slot 78 in the socket member

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72 has a slight arc to allow the pin 76 to move radially outward (when the lever 74 is squeezed) and radially inward (when the lever 74 is released). The slot 78 in the housing 50 is generally C-shaped to include a lock area 82, a folded detent area 84, and a travel area 86 between the two. When the lever 74 is released, as shown in FIG. 11A, the pin 74 can remain located in the lower, lock area 82 and the leg 14 cannot rotate. When the lever 74 is squeezed, the pin 76 rotates into the travel area 86 in the housing slot 80, and the pin 76 rides along this track-like area as the rear leg 14 is pivoted to the fold position. FIG. 8 shows the rear legs 14 in a partially pivoted position. In this position, the locking pin 76 is located in travel area 86. When the rear leg 14 is completely folded, as shown in FIG. 11B, the pin $7\bar{6}$ is free to move to the folded detent area 84 to lock the leg 14 in place. This area 84 has a lead out angled surface 88 that creates a detent or soft lock. Because there is no positive lock in this area 84, the legs 14 can be moved to the in-use position without squeezing the lever 74. The degree of the angled surface 88 will determine the amount of force needed to move the legs 14 to the in-use position. Although the figures show fold mechanisms associated with the rear legs, it is envisioned that, in an alternative swing arrangement, the fold mechanisms could be associated with the front legs.

In addition to the pivoting of the rear legs 14, the swing handle 60 is pivoted during the fold operation. The swing handle 60 is pivoted from an in-use position (one of the open access position, the lift and lock position, and the entertain position) to the storage position, in the direction of arrow B in FIG. 8. The swing handle 60 can be moved to the storage position either before or after folding of the rear legs 14.

Once the swing handle 60 is in the storage position, the seat back 22 and the seat bottom 24 are pivoted to their fold positions. That is, the seat back 22 and the seat bottom 24 are pivoted toward each other until the back and bottom 22, 24 frictionally engage the detents of the respective ribs 304, 306 on the inner surface of the hanger arms 30. The swing handle 60 nests between the seat back 22 and bottom 24 when all three structures are folded. FIG. 9 illustrates the swing 1 in its fully folded position.

If the user wants to carry the folded swing 1, the user can maintain the swing handle 60 in the lift and lock position shown in FIG. 7 and fold the remaining swing structures, including the rear legs 14, the seat back 22, and the seat bottom 24. In this regard, the swing handle assembly operates independently of the remaining fold structures.

FIG. 12 illustrates a swing handle assembly 100 according to an exemplary embodiment of the invention. Such a swing handle assembly 100 is present at each end of the swing handle 60 to mount the swing handle 60 to the frame 10 (shown in FIG. 1). The swing handle assembly 100 includes an end of the swing handle 60 and a handle support structure 110. The handle support structure 110 is positioned within a respective housing 50 (shown in FIG. 1), and it may be integrally molded with the housing 50 or may be attachable to the housing 50. The swing handle 60 is rotationally coupled to the handle support structure 110 such that the swing handle 60 may rotate about a handle rotational axis RA between at least two positions. In addition, the swing handle 60 may include a number of electronic devices 162, as shown in FIG. 1.

The swing handle 60 may comprise a handle portion 112 and a support interface portion 114. The support interface portion 114 is the portion of the swing handle 60 that is attached to the handle support structure 110. The support interface portion 114 is positioned within the respective housing 50 (shown in FIG. 1).

FIG. 13 illustrates the swing handle 60 rotated relative to the handle support structure 110 so that the swing handle 60 is in an open access position. In this position, a central portion of the swing handle 60, which includes the central portion of the handle portion 112, is arranged rearward of the rotational axis of the handle 60. In this application forward of the rotational axis is toward a front of the swing 1 and rearward of the rotational axis is toward the rear of the swing

In this open access position, access to the seat ${f 20}$ is easily $_{10}$ facilitated because the swing handle 60 is out of the way relative to the seat 20, as illustrated in FIG. 14. A child may be easily placed within the seat 20 while the swing handle 60 is rotated rearward. In this open access position, the swing handle 60 is stopped from any further rearward $_{15}$ rotation relative to the handle support structure 110 and the frame 10. The swing handle 60 is stopped relative to the support structure 110 when a handle stop 120 on the handle 60 meets a support stop 122 on the handle support structure

Referring to FIG. 13, the handle stop 120 is located on an outer peripheral wall 124 of the support interface portion 114. The handle stop 120 may be shaped, for example, as a protrusion with a flat edge facing the support stop 122 when in contact with the support stop 122. The support stop 122 25 may be shaped, for example, as a protrusion with a flat edge facing the handle stop 120 when in contact with the handle stop 120. In particular, the support stop 122 may be generally L-shaped with a radial rib 121 of the L facing the handle stop 120. The radial rib 120 extends radially from the axis 30 of rotation. The radial rib 121 contacts the handle stop 120 when the handle stop 120 meets the support stop 122. The support stop 122 may also include a circumferential rib 123 extending in a circumferential direction relative to the axis of rotation. In this respect, the support stop 122 has a dual 35 function: to facilitate positioning of the swing handle 60 in the open access position, as explained above, and to facilitate positioning of the swing handle 60 in the entertain position, as will be explained below.

Rotation of the swing handle 60 relative to the handle 40 support structure 110 to the entertain position is now described with reference to FIGS. 1, 12, and 15. FIG. 15 illustrates the swing handle 60 rotated relative to the handle support structure 110 so that the swing handle 60 is in an entertain position. In this position, a central portion of the 45 swing handle 60 is arranged forward of the rotational axis of the swing handle 60, when the swing handle 60 is arranged as part of the swing. The swing handle 60 and handle support structure 110 are configured so that, when the swing handle swing handle 60 is positioned above and/or in front of a child seated in the swing. Thus, the child would be able to easily view the swing handle 60 and any toys and/or electronic stimuli associated with the handle 60. In this the child. As described further below, the swing handle may include electronic devices 162 (shown in FIG. 1) to provide lights and/or sounds for entertainment.

To maintain the swing handle 60 in the entertain position, the support interface portion 114 includes a detent mecha- 60 nism that frictionally resists rotational motion by the swing handle 60 in a forward or rearward direction relative to the handle support structure 110 and the swing frame 10. The detent mechanism may comprise, for example, one or more swing handle 60 and the handle support structure 110. The other of the support interface portion 114 and the handle

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support structure 110 includes a protrusion, as part of the detent mechanism, arranged such that when the swing handle 60 is rotated in a first direction relative to the handle support structure 110 and the detent and the protrusion meet, the detent frictionally resists rotational motion by the swing handle 60 in the first direction or in a direction opposite to the first direction.

FIG. 15 illustrates an example where the detent 150 is on the support interface portion 114 of the swing handle 60. In this case, the support stop 122 may serve as the protrusion that frictionally resists the detent 150 when the detent 150 and the support stop 122 meet. Alternatively, the protrusion may be other than the support stop 122.

The support stop 122 may be generally L-shaped, as described above with respect to FIGS. 12 and 13. The radial rib 121 of the L extends radially from the center of rotation and acts to stop the handle stop 124, as explained with respect to the open access position of FIG. 13. The radial rib 121 may be angled such that its radially distance from the rotation axis increases along the rotation axis. The circumferential rib 123 of the L extends circumferentially and engages the detent 150 to provide frictional engagement between the support stop 122 and the detent 150 as the detent 150 moves along the support stop 122. The circumferential rib 123 and the height of the detent 150 are set to provide sufficient resistance to rotation to hold the swing handle 60 in the entertain position, but not so much resistance as to make it difficult to rotate the handle out of the entertain position. FIG. 1 illustrates the swing with the handle 60 in the entertain position.

Rotation of the swing handle 60 relative to the handle support structure 110 to the lift position is now described with respect to FIGS. 12, 16, and 17. FIG. 17 illustrates the swing handle 60 rotated relative to the handle support structure 110 so that the swing handle 60 is in the lift position. In this position, a central portion of the swing handle 60 is arranged generally above the rotational axis of the swing handle 60, when the swing handle is arranged as part of the swing. In this lift position, the swing handle 60 is locked relative to the handle support structure 110 and frame 10. The swing 1 may be lifted by grasping the swing handle 60 and lifting. Because the rotational motion of the swing handle 60 is locked relative to the swing frame 10, the swing 1 may be more easily carried without awkwardness otherwise caused by freely swinging motion of the swing frame 10 relative to the swing handle 60.

The swing handle may be locked relative to the swing frame 10 and handle support structure 110 by means of a 60 is in this entertain position, the central portion of the 50 protrusion and matching recess. For example, one of the support interface portion 114 of the swing handle 60 and the handle support structure 110 may include a protrusion, and the other of the support interface portion 114 and the handle support structure 110 may include a recess matched to the regard, the swing handle 60 may include features to entertain 55 protrusion such that, when the protrusion is within the recess, the swing handle 60 is locked relative to the handle support structure 110. The locking mechanism of the protrusion and recess may also incorporate a user-activated

FIGS. 12 and 16 illustrate an example where the protrusion 160 is on an inner peripheral wall 164 of the support interface portion 114 of the swing handle 60, and the recess 162 is on an outer peripheral wall 166 of the handle support structure 110. When the swing handle 60 is rotated such that detents on one of the support interface portion 114 of the 65 the protrusion 160 lines up with recess 162, the handle 60 may be grasped and lifted so that the protrusion 160 enters the recess 162, and further rotational motion of the swing

handle 60 relative to the handle support structure 110, in either rotational direction, is prevented. The locking of the handle 60 relative to the handle support structure 110 may be released by lowering the handle 60 (such as by pushing on the handle 60) relative to the handle support structure 110 to 5 disengage the protrusion 160 from the recess 162. FIG. 17 illustrates the swing with the handle 60 in the lift position.

The lift and lock mechanism described above with the matching protrusion and recess provides a number of advantages. Locking action is transparent to the user with no 10 secondary action required. Moreover, the design uses few moving parts and is easy to assemble. Further, cost effective materials can be used to achieve the desired function.

Rotation of the swing handle 60 relative to the handle support structure 110 to the storage position is now 15 described with respect to FIGS. 12, 20, and 21. FIG. 21 illustrates the swing handle 60 rotated relative to the handle support structure 110 so that the swing handle 60 is in a storage position. In the storage position, the rotational motion of the swing handle 60 relative to the handle support structure 110 need not be stopped, locked, or frictionally resisted by structures on the swing handle 60 and/or handle support structure 110, because the relative rotation is prevented by nesting of the swing handle 60 between the seat back 22 and the seat bottom 24 of the seat 20. The swing handle 60 may be nested between the seat back 22 and the seat bottom 24 of the seat 20, when the swing is in a folded position.

According to one embodiment of the invention illustrated in FIG. 12, the handle 60 and handle support structure 110 include structure to allow the handle 60 to be snapped onto the handle support structure 110, and thereafter the handle 60 is rotationally fixed to the handle support structure 110. In this regard, the outer peripheral wall 166 of the handle support structure 110 includes a snap finger 180. When the handle 60 is assembled to the handle support structure 110 such that the inner peripheral wall 164 of the support interface section 114 passes over and past the snap finger 180, the snap finger 180 extends radially outward and beyond a lip 182 of the inner peripheral wall 164. This extension of the snap finger 180 beyond the lip 182 prevents the handle 60 from being slid off of the handle support structure 110.

FIGS. 18 and 19 illustrate another aspect of the invention 45 wherein electrical wiring 161 extends from the handle portion 112 of the swing handle 60 into and through the support structure 110 so that electronic devices 162 (see FIG. 1) on the handle portion 112 may be powered by a power supply not in the handle portion 112, but in one of the 50 arc length of the generally arc shaped contact 172 deterhousings 50.

The wiring 161 extends into a cavity 164 within the handle portion 112 to the electronic devices 162 on the handle portion 112. The electronic devices may be, for example, light producing electronic devices and/or sound 55 electrical devices 162 may need to operate only over a producing electronic devices. For example, if the electronic devices 162 are for the entertainment of a child in the swing, one or more of the electronic devices 162 may be a colored light shaped as a pleasing design for a child, such as a star or a cat. The electronic devices 162 may also produce sounds 60 may need to operate only over a rotational range where the instead of, or in addition to, light. For example, if the electronic device is a colored light shaped as a cat, the device may also produce a "meow" sound. One or more of the electronic devices 162 may also produce sounds such as music, for example.

The support interface portion 114 may include an outer peripheral wall 170 adjacent the handle portion 112. In order 10

to pass the wiring 161 from the cavity 164 of the handle portion 112 to the support interface portion 114 of the handle 60, the outer peripheral wall 170 may include a slot 171. The slot 171 allows for an electrical connection between the handle portion 112 and the interface portion 114. An electrical connection or contact between the support interface portion 114 and the handle support structure 110 may be implemented by means of at least one moving contact

Beneficially the moving contact assembly allows electronics to be powered in a movable handle, i.e., the handle 60, through wiring passing through a rotating joint, i.e., the joint of the support interface portion 114 and the handle support structure 110.

The moving contact assembly may comprise a generally arc shaped contact 172 on the handle support structure 110 and at least one spring contact 174 on the support interface portion 114. The spring contacts 174 are adapted to electrically contact the generally arc shaped contact 172 as the swing handle 60 rotates relative to the handle support 20 structure 110.

The generally are shaped contact 172 may comprise a printed circuit board or conductive ink formed on a surface of the handle support structure 110, for example. If the generally are shaped contact 172 comprises a printed circuit board, the handle support structure 110 may comprise a board mounting slot, so that the printed circuit board may be fixedly attached to the handle support structure 110 via the board mounting slot by snapping into the slot. Alternately the printed circuit board may be fixedly attached to the handle support structure 110 by screws or glue. The wiring 160 electrically contacts the generally arc shaped contact 172 via spring contacts 174.

The spring contacts 174 may be formed of any appropriate material, and may be, for example, formed of a sheet 35 metal stamping, conductive plastic, or graphite, for example.

The spring contacts 174 may pass through respective slots of the at least one slot 178 on the support interface portion 114. The wiring 161 may we attached to the support interface portion 114 by wrapping the wiring 161 around respec-40 tive support posts 301. The spring contacts 174 may be attached to the wiring 161 using a contact snap 180 attached to the support interface portion 114. Power supply wiring (not shown) may then extend from the generally arc shaped contact 172 to the power supply 52 (shown in FIG. 1).

The moving contact assembly comprising the generally arc shaped contact 172 and the spring contacts 174 provides an electrical contact between the generally are shaped contact 172 and the spring contacts 174 as the swing handle 60 is rotated relative to the handle support structure 110. The mines the rotational range over which electrical contact is maintained between the generally arc shaped contact 172 and the spring contacts 174, and thus the range over which power is supplied to the electronic devices 162. Because the limited rotational range of the handle 60, limiting the arc length of the generally arc shaped contact 172 is possible, and the limited size of the generally arc shaped contact 172 may beneficially reduce its cost. The electrical devices 162 swing handle 60 rotates over a certain angle forward and rearward of the entertain position, for example. In one embodiment, the position and arc length of the generally arc shaped contact 172 is configured so that the electrical 65 devices 162 work at the lift position and at ±60 degrees from the lift position, where +60 degrees includes the entertain position.

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As an alternative, the swing handle 60 itself may contain a battery support structure for containing batteries and providing power to the electronic devices 162 on the swing handle 60. In this case, the swing handle 60 need not include wiring to the power supply 52 within the housing 50.

The preferred embodiments have been set forth herein for the purpose of illustration. This description, however, should not be deemed to be a limitation on the scope of the invention. Various modifications, adaptations, and alternafrom the claimed inventive concept. The true scope and spirit of the invention are indicated by the following claims.

What is claimed is:

- 1. A swing comprising:
- a swing frame;
- a swing seat;
- at least one hanger arm connecting the swing seat to the swing frame;
- a motor mounted to the swing frame to drive the motion 20 of the at least one hanger arm and the swing seat; and
- a swing handle rotationally coupled to the swing frame and extending over the swing seat, wherein the swing handle may rotate about a handle rotational axis between at least two positions.
- 2. The swing of claim 1 wherein the at least one hanger arm is rotationally coupled to the swing frame and supports
- 3. The swing of claim 1, wherein the at least two positions include a lift position wherein a central portion of the swing 30 handle is arranged above the rotational axis.
- 4. The swing of claim 1, wherein the at least two positions include an open access position wherein a central portion of the swing handle is arranged rearward of the rotational axis.
- 5. The swing of claim 1, wherein the at least two positions 35 include an entertain position wherein a central portion of the swing handle is arranged forward of the rotational axis.
- 6. The swing of claim 1, wherein the swing frame comprises a handle support structure, and wherein the swing handle is rotationally coupled to the swing frame at the 40 handle support structure.
- 7. The swing of claim 6, wherein the handle support structure includes a snap finger and the swing handle includes a lip, such that, when the swing handle is slid onto the handle support structure, the snap finger engages the lip 45 to prevent the swing handle from being slid off the handle support structure.
- 8. The swing of claim 6, wherein one of the swing handle and the handle support structure includes a protrusion, and the other of the swing handle and the handle support 50 structure includes a recess matched to the protrusion such that, when the protrusion is within the recess, the swing handle is locked to prevent rotation relative to the handle support structure.
- 9. The swing of claim 6, wherein one of the swing handle 55 and the handle support structure includes a detent, and the other of the swing handle and the handle support structure includes a protrusion arranged such that, when the swing handle is rotated in a first direction relative to the handle support structure and the detent and the protrusion meet, the 60 detent frictionally resists rotational motion by the swing handle in the first direction or in a direction opposite to the first direction.
- 10. The swing of claim 9, wherein the swing handle comprises a handle stop and the handle support structure 65 comprises a support stop arranged such that, when the handle is rotated in a first direction relative to the handle

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support structure and the handle stop and support stop meet, further rotational motion of the swing handle relative to the handle support structure is prevented.

- 11. The swing of claim 10, wherein the handle support 5 structure includes one rib extending radially from the handle rotational axis and another rib extending circumferentially relative to the handle rotational axis, and wherein the protrusion comprises the rib extending circumferentially.
- 12. The swing of claim 6, wherein the swing handle tives may occur to one skilled in the art without departing 10 comprises a handle stop and the handle support structure comprises a support stop arranged such that, when the handle is rotated in a first direction relative to the handle support structure and the handle stop and support stop meet, further rotational motion of the swing handle relative to the 15 handle support structure is prevented.
 - 13. A swing comprising:
 - a swing frame;
 - a swing seat;
 - at least one hanger arm connecting the swing seat to the swing frame;
 - a motor mounted to the swing frame to drive the motion of the at least one hanger arm and the swine seat; and
 - a swing handle rotationally coupled to the swing frame such that the swing handle may rotate about a handle rotational axis between at least two positions
 - wherein the at least two positions include a lift position wherein a central portion of the swing handle is arranged above the rotational axis and wherein the swing handle is locked to prevent rotation of the swing handle relative to the swing frame.
 - 14. A swing comprising:
 - a swing frame;
 - a swing seat;
 - at least one hanger arm connecting the swing seat to the swing frame;
 - a motor mounted to the swing frame to drive the motion of the at least one hanger arm and the swing seat; and
 - a swing handle rotationally coupled to the swing frame such that the swing handle may rotate about a handle rotational axis between at least two positions
 - wherein the at least two positions include an open access position wherein a central portion of the swing handle is arranged rearward of the rotational axis and wherein the swing handle is stopped from any further rearward rotation of the swing handle relative to the swing frame.
 - 15. A swing comprising:
 - a swing frame;
 - a swing seat;
 - at least one hanger arm connecting the swing seat to the swing frame;
 - a motor mounted to the swing frame to drive the motion of the at least one hanger arm and the swine seat; and
 - a swing handle rotationally coupled to the swing frame such that the swing handle may rotate about a handle rotational axis between at least two positions
 - wherein the at least two positions include an entertain position wherein a central portion of the swing handle is arranged forward of the rotational axis and wherein a detent mechanism frictionally resists rotational motion by the swing handle in a forward or rearward direction relative to the swing frame.
 - 16. A swing comprising:
 - a swing frame;
 - a swing seat; and